

Article A Study on the Lateral Ultimate Strength and Collapse Modes of Doubly Curved Stiffened Plates

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Abstract: Bows and stems are often subjected to wave slamming loads. Stiffened plates with curvatures in both longitudinal and transversal directions are the basic members of these structures. As a result, it is important to investigate the lateral ultimate strength of the doubly curved stiffened plates. In this study, the non-linear finite element method (NFEM) is selected to investigate the collapse modes and lateral ultimate strengths of the doubly curved stiffened plates. Additionally, the influences of curvature and geometrical properties on the collapse modes and ultimate strengths of doubly curved stiffened plates are investigated. In the NFEM analysis, a series of numerical simulations, covering different aspect ratios, curvatures, and structural scantlings are performed. Different collapse modes of doubly curved stiffened plates under lateral loading cases are observed. The relationships between the collapse modes and the geometrical properties are then discussed based on the numerical results. Moreover, the results also show that larger curvatures along the stiffeners and stronger stiffeners contribute more to the lateral ultimate strengths. Subsequently, an empirical formula is derived and verified for predicting the lateral ultimate strength of the doubly curved stiffened plates. The results of the empirical formula match well with numerical calculations.

Keywords: stiffened plates; longitudinal and transversal curvatures; lateral ultimate strength; collapse modes; empirical formula

1. Introduction

Complicated curved surfaces exist in ship structures, e.g., the bow, stem, and hull surfaces exhibit varying curvatures. Consequently, stiffened plates with curvatures in both longitudinal and transversal directions are employed in these structures. In recent years, there has been increasing interest from both industry and academia in the structural ultimate strength and reliability of bows and stems.

Ultimate strengths of bows and stems are very important in the evaluation of ship structures, especially in ship-ice collision and wave slamming cases. Yang et al. [1] researched the dynamic behaviors of the large container ship's bow structures subjected to slamming pressures; a safety margin evaluating the safety performance of large container ships under a slamming pressure coefficient was presented. Shabani et al. [2] investigated the slamming loads and kinematics during bow entry events and derived an advanced central bow designing method. In such cases, lateral loads with large amplitudes are applied to doubly curved stiffened plates which constitute structures. The plates face the risk of global buckling, which can result in the collapse of the entire structure. Therefore, it is crucial to conduct further research on the ultimate strength issues associated with such scenarios. Moreover, doubly curved stiffened plates are the basic units of ship structures with complicated curved surfaces, of which the collapse mechanisms are not exactly the same as with flat stiffened plates or singly curved stiffened plates. Related research into doubly curved stiffened plates has seldom been reported.



Citation: Guo, G.; Cui, J.; Wang, D. A Study on the Lateral Ultimate Strength and Collapse Modes of Doubly Curved Stiffened Plates. J. *Mar. Sci. Eng.* **2023**, *11*, 2315. https://doi.org/10.3390/ jmse11122315

Academic Editor: Joško Parunov

Received: 5 November 2023 Revised: 30 November 2023 Accepted: 5 December 2023 Published: 7 December 2023



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Research on the ultimate strengths and collapse mechanisms of flat stiffened plates serves as a valuable reference for studying curved stiffened plates. Numerous empirical formulae have been developed to conveniently obtain the ultimate strength of the plates or stiffened plates. Liu and Zhang [3] analyzed the influence of the rotational stiffness, aspect ratio, and initial deflection on the ultimate strengths of plates subjected to axial compressions, and a modified empirical formula containing the above factors was proposed. Hayward and Lehmann [4] presented a new formula of the load-bearing capacity of plates under biaxial loads, which can capture the influences of the plate slenderness and aspect ratio. Kim et al. [5] reviewed the well-known formula, predicting the ultimate strength of stiffened plates by comparing it with analytical and FEA results, and proposed the ultimate strength empirical formula without considering the fluctuation behavior. Li et al. [6,7] presented an adapted algorithm to predict the stiffened plates' collapse progress, which extends the predicting capacity of the elastic stiffness and ultimate and post-ultimate features. The ultimate strength of stiffened plates subjected to the combined longitudinal compressive stress and lateral pressure has also been studied, with corresponding numerical analysis, empirical formulae, and experiments. Yao et al. [8] explored the effect of the boundary conditions on the ultimate strength of stiffened panels under the combined action of longitudinal and lateral loads.

Many investigations on collapse behaviors of plane stiffened plates have also been performed, through which several typical collapse modes, deformation mechanisms, and some recommending modeling techniques have been explored. Paik [9] explored six kinds of collapse modes for stiffened plates under different load cases and derived a series of empirical formulae for combined load cases. Ma et al. [10] explored the influence of lateral loads on ultimate strengths and collapse modes of stiffeners under combined load cases and provided a useful explanation about different collapse modes of stiffeners. Xu et al. [11] performed a series of NFEM analyses and investigated the influence of boundary conditions on collapse behaviors of the stiffened plates under different load cases. Li et al. [12,13] investigated the deformation behaviors of continuous plates under combined biaxial loads and lateral pressure. The loading components include both constant loads and cyclic loads.

By taking transversal curvature into account, many studies on singly curved stiffened plates have been conducted. Cui et al. [14] investigated the ultimate strength and collapse behaviors under different load cases of single-curved stiffened plates from cargo containers and derived a series of empirical formulae for predicting the axial ultimate strength of the single-stiffened plates. Park et al. [15] researched the fundamental buckling behaviors of cylindrically curved plates subjected to axial loading and investigated the effects of the curvature, magnitude of initial imperfection, slenderness ratio, and aspect ratio on the characteristics of the buckling and post-buckling collapse behavior of cylindrically curved plates. Seo et al. [16] researched the influence of curvatures on the buckling/postbuckling characteristics and collapse behaviors of singly curved stiffened plates under axial compression and then derived relevant formulae on axial strengths. Park et al. [17] performed a series of elastic-plastic large deflection analyses on singly curved stiffened plates to clarify the fundamental behaviors of cylindrically curved plates under axial compression and lateral pressure. A double beta formula considering secondary buckling behaviors was derived based on the results. Cho et al. [18] performed an experiment on singly curved stiffened plates and investigated the influence of the curvature effect on the ultimate strengths of the single-curved stiffened plates. Research on the collapse modes and ultimate strengths of flat or singly curved stiffened plates has laid a foundation for studying doubly curved stiffened plates. The methodologies, concepts, and ultimate strengths' influencing factors involved in these studies are instructive.

In this study, the collapse behaviors and ultimate strengths of doubly curved stiffened plates under lateral loading cases are investigated. Considering different kinds of ship structures, the influences of varying types of stiffeners, longitudinal and transversal curvatures, and geometric scantlings on the ultimate strengths and collapse modes are taken into account. Based on above influential factors studies, a prediction formula for lateral ultimate strengths of such structures has been derived and verified, resulting in good accuracy with simple form. The results from this research can be helpful for the understanding of doubly curved panels' ultimate strength and collapse behavior as well as its contribution to the ultimate strengths of ship structures with complicated curved surfaces.

2. Methodology

In this study, the geometrical parameters of doubly curved stiffeners are firstly obtained according to the data from different ships. The NFEM models of the doubly curved with different slenderness ratios are established in the next step. Then, the lateral ultimate strengths and collapse modes of the models are obtained via the NFEM. Finally, an empirical formula of the doubly curved stiffened plates' lateral ultimate strength is derived.

2.1. Description of Doubly Curved Stiffened Plate

Doubly curved stiffened plates make up the complex surfaces of bows and stems, and they contribute to the strengths and the loading capacities of the structures. Such stiffened plates have curvatures in both the longitudinal and transversal directions, and the plates form complicated curved surfaces with varying curvatures. The stiffened plates in this study are selected as single span and single bay models. According to the recommendation of ISSC (2012) [19], such a model extent is acceptable. As such, stiffened plates are supported by transversal and longitudinal primary supporting members (PSMs), and, in the case of lateral loading, the boundary conditions of the stiffened structures at the PSMs can be regarded as clamped supported. Considering the background of this study, several stiffened plates derived from the bows and stems of an icebreaker, an oil tanker, and a cargo container are selected as references models. Models with different curvatures are set to explore the influence of curvature on the ultimate strengths of doubly curved stiffened plates. The structural settings of the models are shown in Figure 1, and the original geometric parameters can be found in Table 1. And the parameters appear in this research are shown in Nomenclature Section, where units for geometry and stress are mm and MPa, respectively. The detailed structural settings of the models in this research can be found in Appendix A.



Figure 1. Structural settings of typical doubly curved stiffened plates: (**a**) geometric model; (**b**) NFEM model.

As shown in Figure 1a, *a* denotes the longitudinal span, *b* denotes the spacing between adjacent stiffeners, R_L denotes the curvature radius along the stiffeners, and R_T denotes the curvature radius along the direction that is perpendicular to the stiffeners. To illustrate the curvature and its influence on the ultimate strengths, the curvatures in longitudinal and transversal directions of all the models are represented by groups of dimensionless terms $\theta_L = a/R_L$ and $\theta_T = b/R_T$, which are shown in Figure 2. All the stiffeners are equipped with a 'T' type section, which is depicted in Figure 3.

Ship	<i>R_L</i> (mm)	<i>R_T</i> (mm)	a (mm)	b (mm)	Stiffener	Aspect Ratio	σ _Y (MPa)	Number of Stiffeners
Icebreaker	14,523	40,306	4000	350	Tee bar, T360 \times 20 + 90 \times 20	11.42	355	5
Icebreaker	17,075	48,849	4000	350	Tee bar, T360 \times 20 + 90 \times 20	11.42	355	7
Icebreaker	5719	45,560	2375	350	Tee bar, T360 \times 20 + 90 \times 20	6.78	355	5
Icebreaker	5475	54,020	2375	350	Tee bar, T360 \times 20 + 90 \times 20	6.78	355	7
Container ship	74,243	4770	2400	650	Tee bar, T280 \times 11 + 120 \times 20	3.69	355	3
Container ship	13,104	7198	3044	675	Tee bar, T320 \times 12 + 120 \times 20	4.50	355	3
Icebreaker	11,500	59 <i>,</i> 500	2375	350	Tee bar, T360 \times 20 + 90 \times 20	6.78	355	5
Oil tanker	26,500	37,500	3200	800	Tee bar, T420 \times 10 + 120 \times 16	4	355	5
Oil tanker	41,600	27,200	3200	800	Tee bar, T420 \times 10 + 120 \times 16	4	355	5

Table 1. Structural settings of the original doubly curved stiffened plates models for this study.



Figure 2. The curvature radiuses of the stiffened plate models.



Figure 3. The dimensions of the section of the stiffener.

As there is curvature in the longitudinal or transversal direction, Seo et al. [16] investigated the singly curved stiffened plate and concluded that the ultimate strengths of such plates are related to the slenderness of flat plates with identical geometric dimensions. Therefore, the structural strengths of doubly curved stiffened plates can be evaluated by the slenderness ratios λ and β , where λ denotes the column slenderness ratio of stiffener with attached plating and β denotes the slenderness ratio of plating between stiffeners, which are expressed by Equation (1):

$$\beta = \frac{b}{t_p} \sqrt{\frac{\sigma_Y}{E}}$$

$$\lambda = \frac{a}{\pi r} \sqrt{\frac{\sigma_Y}{E}}$$

$$r = \sqrt{\frac{I}{A}}$$
(1)

To clarify the influence of initial imperfections on lateral ultimate strengths, a curved stiffened plate with transversal curvature identical to model 'R5' is established. The 'buckling' mode initial imperfection recommended by ISSC [19] and the 'thin horse' mode initial imperfection conducted by Yao et al. [20] are applied to the model, respectively. The modes of the initial deflections are depicted in Figure 4 and Equations (2)–(5). As shown in Figure 5, the lateral strengths of models with different geometrical imperfections are similar. Moreover, the lateral ultimate strength does not significantly decrease with the occurrence of the initial deflections, and the differences in the ultimate strengths between models with and without initial deflection are less than 5%. Consequently, the influence of the initial deflections can be considered reasonable.



Figure 4. The modes' initial imperfections. (a) Buckling mode; (b) thin horse mode. Scale: 40.



Figure 5. Influence of initial imperfections on ultimate strengths.

The column type deformation of a stiffened plate:

$$v_{oc} = B_0 \sin\left(\frac{\pi z}{a}\right) \sin\left(\frac{\pi x}{B}\right) \tag{2}$$

where B_0 denotes the magnitude of column type mode and is recommended as 0.0015a. The initial tripping distortion of a stiffener:

$$v_{oc} = B_0 \sin\left(\frac{\pi z}{a}\right) \sin\left(\frac{\pi x}{B}\right) \tag{3}$$

The buckling mode initial deflection of local plating:

$$v_{opl} = A_0 \sin\left(\frac{m\pi z}{a}\right) \sin\left(\frac{\pi x}{b}\right) \tag{4}$$

The thin horse mode initial deflection of local plating:

$$v_{opl} = \left| \sum_{k=1}^{11} A_{0k} \sin\left(\frac{k\pi z}{a}\right) \sin\left(\frac{\pi x}{b}\right) \right|$$
(5)

where A_{0k} are determined according to the empirical value derived by Yao et al. [20], and is shown in Table 2, and the maximum magnitude A_0 is the same as the previous buckling mode.

Table 2. Coefficients of initial deflection magnitudes.

A_{01}/A_0	A_{02}/A_0	A_{03}/A_0	A_{04}/A_{0}	A_{05}/A_0	A_{06}/A_0	A_{07}/A_0	A ₀₈ /A ₀	A_{09}/A_0	A ₁₀ /A ₀	A_{11}/A_0
1.1439	-0.0677	0.3385	0.0316	0.1579	-0.0149	-0.0043	0.0008	0.0039	-0.0002	-0.0011

In shipyards, the manufacturing of doubly curved plates with complicated curvatures is achieved through the line heating technique. Consequently, the existing empirical formulae for initial imperfections such as 'buckling' and 'thin horse' are not suitable for such plates. Moreover, in actual conditions, residual stress will shake down after multiple loading and de-loading processes; thus, the influence of residual stress is negligible in a real ship structure. The calculations performed above have proved that the influence of the initial deflection is neglectable, and there are few data about the initial imperfections that are applicable for doubly curved stiffened plates. Therefore, the influence of the initial deflections and residual stress is not considered in this study.

2.2. NFEM Analysis

The computer code ABAQUS is considered as the numerical analyzing method in this study. The type of elements is chosen as S4R, which simulates the shell and possesses four nodes and six degrees of freedom. In the NFEM models, the nonlinearities of material and geometry are all taken into account. The geometrical nonlinearity is performed by the using the 'NLGEOM' option in the code. This option takes large deformations and the nonlinearity between displacements and strains into account. And the nonlinearity of the material is performed by considering the yielding strength in the constitutive relationship of the material. Material nonlinearity and geometric nonlinearity must be considered in research, while initial imperfection is not a mandatory input; meanwhile, there is no relevant data available for doubly curved stiffened plates, so it is not considered in this study. The property of steel is defined with the ideal elastic-plastic model (the material has no plastic strain, and the nonlinearity is realized by yielding strength, σ_Y), which is illustrated in Figure 6.



Figure 6. The ideal elastic-plastic constitutive relationship for the steel.

Additionally, the implicit static method is applied during the calculation. As the size of the mesh has an important impact on the speed and results of the calculation, to keep a balance between the accuracy and efficiency of the calculation, an appropriate size of the mesh should be confirmed. The mesh sizes are selected as 10 mm, 15 mm, 20 mm, 25 mm, and 45 mm, respectively. As shown in Figure 7, the ultimate strength exhibits a stable state when the element sizes are within 10 mm and 20 mm. As a result, an element size of 20 mm was used in the finite element model to ensure relatively accurate results and higher calculation efficiency. Moreover, to justify the reliability of the NFEM, we also performed a comparison calculation on an axial ultimate strength problem of the standard stiffened plates conducted by ISSC 2012 [19], and the relevant information and results are shown in Table 3. The error between the results of the NFEM and ISSC is within 2.5%, signifying that the NFEM employed in this study is reliable.



Figure 7. Results of the mesh sensitivity analysis.

Table 3. Results of the benchmark calculation.

σ_Y (MPa)	<i>t_p</i> (mm)	Stiffeners' Parameters (mm)	Result (MPa)	Result of ISSC (MPa)	Error
313.6	15	$\begin{array}{c} 580 \times 15/150 \times 20, \\ \text{Tee Bar} \end{array}$	232.52	227.05	2.4%

2.3. Boundary Conditions and Load Applications

The quantity of longitudinal and transversal primary supporting members in bow or stem structures is generally larger than those in other parts; therefore, doubly curved stiffened plates are subjected to stronger constraints than stiffened plates in other parts. Additionally, the deformations of the shell, when subjected to uniformly distributed lateral loads, are symmetric in longitudinal or transversal direction. Consequently, in the investigation of the lateral ultimate strength of the doubly curved stiffened plates, the clamped constraint is applied on the boundaries of the numerical model, restricting all the translations and rotations. The lateral loads are applied on the outer surface without stiffeners in the form of a uniformly distributed normal load. The settings of load and boundary conditions are shown in Figure 8.



Figure 8. NFEM settings. (a) Boundary conditions; (b) lateral loading.

3. Results and Discussion

Based on the NFEM, a series of calculations have been performed on typical doubly curved stiffened plate models designed according to corresponding ships. The following results have been obtained by analyzing the target models with 144 different structural dimensions. The variations in the curvature radius, plate thickness, and stiffeners' sections are considered in the design of these numerical models. According to the structural designs of target ships and the information from Zhang [21], the column slenderness of the model ranges from 0 to 1.5 and the plate slenderness ranges from 0 to 3. To describe the geometrical properties conveniently, the stiffeners are named in the form of 'Curvature radius-Plate thickness-Stiffener'. Nine groups of curvature radiuses are represented by 'R1'-'R9'. In each group, four types of plate thickness and stiffeners' sections are selected, denoted as 'T1'-'T4' and 'S1'-'S4', respectively, and the corresponding number represents a specific type. For example, 'R2' denotes these models have the same longitudinal and transversal curvature radius as those in group 2, while 'T1' or 'S1' denotes the same setting as the original model. Similarly, the model 'R7T2S3' signifies that the plate thickness and stiffeners' properties differ from those of the original models.

3.1. The Determination of the Lateral Ultimate Strength

In this study, the lateral ultimate strength of the doubly curved stiffened plate is determined by identifying the lateral load at the inflection point of the lateral load-central displacement curve. The magnitude displacement of the central point is selected as the response of the structure in the lateral loading cases. Figure 9b illustrates the lateral load-magnitude displacement curve for each case, where four points are selected to define two straight lines. These lines represent different parts of the curve; namely, the linear loading stage before buckling occurs and the post-buckling stage with large deformations. The lateral load at the intersection of these lines is considered as the lateral ultimate load, measured in MPa. The lateral ultimate strength and the structural response are normalized by σ_U/σ_Y and w/t_p , respectively, where σ_Y denotes the yielding stress of the material and u_c denotes the normal displacement of the central point of the model.



Figure 9. The definition of the lateral ultimate strength. (**a**) The central point of the model (the red highlighted node); (**b**) the determination of the lateral ultimate load.

3.2. Collapse Modes and Ultimate Strength of Doubly Curved Stiffened Plates under Lateral Pressure Loading

3.2.1. The Lateral Ultimate Strength of the Doubly Curved Stiffened Plates

The lateral load—central displacement curves of model R1 are shown in Figure 10. The column slenderness ratios of model R1 range from 0.368 to 0.684, while the plate slenderness ratios range from 0.605 to 1.816. The thicknesses of plates are set as 24 mm, 14 mm, 11 mm, and 8 mm, respectively, and are labeled as 'T1'-'T4'. Initially, the displacement of the central point increases slowly as the lateral load increases, indicating that the lateral load applied to the stiffened plate is within its loading capacity. However, once the lateral load exceeds the ultimate lateral strength, even a small increment in lateral load leads to rapid growth in the normal displacement of the central point, indicating the loss of the loading ability and collapse of the stiffened plate. Figure 10 shows that when λ is set constant, the lateral ultimate strengths decrease significantly for the stiffened plates with thinner plates. Models with thickness 'T4' exhibit the lowest lateral ultimate strength. It can be inferred that for the doubly curved stiffened plates with stronger stiffeners, the thicknesses of the plates exert a significant influence on the lateral ultimate strength.



Figure 10. The lateral ultimate strength-displacement curve of model R1. (**a**) Models with stiffener 'S1'; (**b**) models with stiffener 'S2'; (**c**) models with stiffener 'S3'; (**d**) models with stiffener 'S4'.

The lateral ultimate strengths of models 'R1'–'R4' are depicted in Figure 11. The distance between adjacent stiffeners and the plate slenderness ratios of models 'R1'–'R4' are identical. Models 'R3' and 'R4' have smaller longitudinal curvature radiuses, resulting in significantly larger longitudinal curvature angles (0.415 rad for 'R3'; 0.434 rad for 'R4'; 0.275 rad for 'R1'; and 0.234 rad for 'R2'). Figure 11 shows that as the stiffeners become weaker, the lateral ultimate strengths of the doubly curved stiffened plates decrease. The relationship between lateral ultimate strength and column slenderness is inversely proportional. An interesting phenomenon is observed in cases with different plate thicknesses: the lateral bearing capacity increases significantly as the curvature angle decreases. On one hand, the curves for 'R3' and 'R4' are above those for 'R1' and 'R2'. On the other hand, the lateral bearing ability of models 'R3' and 'R4' are stronger than those of models 'R1' and 'R2' even if the column slenderness ratios are significantly larger. As the curvature angle of the stiffener increases, the shapes of the stiffeners. Consequently, the lateral bearing capacities increase significantly.



Figure 11. The lateral ultimate strengths of models R1–R4: (**a**) $\beta = 0.605$, (**b**) $\beta = 1.038$, (**c**) $\beta = 1.321$, and (**d**) $\beta = 1.816$.

In addition, the influence of low temperature has been considered in this study. Wang et al. [22] conducted a uniaxial tensile experiment on the marine steel selected for this study to investigate the effect of low-temperature conditions on the steel's properties. The results indicate that low temperature increases the elastic modulus and yielding strength, and the ideal elastic-plastic model is still applicable under these conditions. According to the outcomes of the experiment, the elastic modulus is set as 228,000 MPa and the yielding strength is set as 373 MPa (to simulate the steel at -20 °C). Figure 12 shows the lateral strengths of model 'R4T1S1' under 20 °C and -20 °C. The shape difference between the two curves is not significant, while the lateral ultimate strength of the model under -20 °C is higher. And the difference between the two cases is 4.13%. As the temperature selected in the calculation is similar to the actual working conditions of ice breakers, it can be concluded that the lateral ultimate strength of the plates will increase slightly in the actual low-temperature environment. Consequently, results from models employed in this paper are conservative.



Figure 12. The lateral ultimate strengths of model R4T1S1 (at different temperatures).

The influence of the boundary condition has also been researched. The simply supported boundary condition is set on the edges of model 'R4S1'. The ultimate strengths under different boundary conditions are shown in Figure 13. There is a difference of around 10% between the results from models with different boundary conditions. From Figure 10, it can be concluded that the clamped boundary condition results in much larger ultimate strengths. However, since the members in the bows and stems are stronger, the results of the models with clamped boundary conditions are still referential.



Figure 13. The lateral ultimate strengths of model 'R4S1' (with different boundary conditions).

3.2.2. Typical Collapse Modes of the Doubly Curved Stiffened Plates

The stress distribution and collapse mode of models 'R1T1S1' and 'R2T1S1' are shown in Figure 14a–d. From the picture, it can be inferred that for stiffened plates with longitudinal curvature angles around 0.2, yielding and severe deformation mainly occur in the center of the plate, the upper part of the mid-span of the stiffeners, and the areas around the endings of the stiffeners. Figure 10a,b show that under lateral loading, the yielding firstly occurs around the endings of the stiffeners, and the yielding of the plates and the mid-span of the flanges occurs suddenly after the collapse around the endings of the stiffeners. Figure 15a,b demonstrate that under lateral loading, yielding first occurs around the endings of the stiffeners. The plates and mid-span of the flanges suddenly yield after the stiffeners collapse around the endings. Such a phenomenon indicates that the



plate and the stiffener buckle from the peak of the curved surface simultaneously, and the corresponding collapse mode of doubly curved stiffened plates can be described as global collapse originating from the central point.

Figure 14. The collapse modes of models R1T1S1 and R2T1S1: (**a**) the deformation and stress distribution at the ultimate state of model 'R1T1S1'; (**b**) the front view of model 'R1T1S1'; (**c**) the deformation and stress distribution at the ultimate state of model 'R2T1S1'; (**d**) the front view of model 'R2T1S1'. Scale: 10.



Figure 15. The deformations and stress distributions of model R1T1S1 before the ultimate state: (a) the global view of model 'R1T1S1'; (b) the front view of model 'R1T1S1'. Scale: 10.

The collapse modes and stress distributions of the models with larger curvature angles are shown in Figure 16. In Figure 16b,f, besides global deformation, the local buckling of plates is also observed. In comparing with the phenomena shown in Figure 14, the amplitude of the global collapse decreases, whereas the area of yielding in the plate expands. Figure 17 shows the stress distribution in model 'R3T1S1' and model 'R4T1S1' before reaching the ultimate state. It is evident that the global buckling and tripping of the stiffeners occur simultaneously. These phenomena signify that the plates are subjected to stronger constraints in the normal direction, leading to increased participation of the stiffened plates in the loading capacity. In model 'R3T1S1' and model 'R4T1S1', larger curvature angles enhance the lateral bearing capacity of the stiffeners; hence, stronger



normal constraints are applied to the plates. The lateral bearing capacity of the stiffened plates is finally enhanced.

Figure 16. The collapse mode of models 'R3' and 'R4': (**a**) the deformation and stress distribution at the ultimate state of model 'R3T1S1'; (**b**) the front view of model 'R3T1S1'; (**c**) the deformation and stress distribution at the ultimate state of model 'R3T1S3'; (**d**) the front view of model 'R3T1S3'; scale: 15. (**e**) The deformation and stress distribution at the ultimate state of model 'R4T1S1'; (**f**) the front view of model 'R4T1S1'; (**g**) the deformation and stress distribution at the ultimate state of model 'R3T1S3'; (**h**) the front view of model 'R3T1S3'; (**h**) the front view of model 'R3T1S3'; (**h**) the front view of model 'R3T1S3'; scale: 15.



Figure 17. The stress distribution before the ultimate state in models 'R3' and 'R4': (**a**) Model 'R3T1S1'; (**b**) Model 'R4T1S1'; (**c**) Model 'R3T1S3'; (**d**) Model 'R4T1S3'; scale: 15.

The collapse modes and stress distribution in models 'R3' and 'R4' with weaker stiffeners are illustrated in Figure 16c,d,g,h. In these models, the global collapse occurs during the loading process. It is because the lateral bearing capacity of stiffeners decreases as the column slenderness ratio increases, resulting in the global buckling collapse mode. As shown in Figure 17c,d, the yielding happens in the areas around the endings of the stiffeners and the center of the plates before reaching the ultimate state. This phenomenon indicates that larger curvature angles also enhance the lateral bearing capacity of the plates, and the yielding of the plates occurs earlier, which differs from the collapse mode observed in models 'R1' and 'R2'.

The deformation behaviors and stress distributions of model 'R1S1' at the ultimate state are shown in Figure 18. It is observed that the thickness of the plate gradually decreases, leading to a more noticeable buckling of local plates. Particularly, the most severe local buckling occurs in the mid-span of the plates. As the plate slenderness ratio increases, the local buckling of the plates becomes significant, and the collapse mode converts to the combination of global buckling of the stiffened plates and the local buckling of the plates when the plate slenderness ratio becomes larger.

The deformation behaviors and stress distributions of models 'R3S1' are shown in Figures 19 and 20. In models with larger curvature angles along the stiffeners, the local buckling of plates becomes more significant as the plate slenderness ratio increases. As the plates become thinner, the local buckling of the plates becomes more severe, and the buckling of plate happens earlier. Consequently, the dominant collapse mode converts into the local buckling of the plates.

The deformation behaviors and stress distribution of models with larger slenderness ratios of plates are shown in Figure 21. In contrast to the models 'R1' and 'R2' taken from an ice breaker, these models have larger plate slenderness ratios. Another collapse mode, which is different from the models discussed above, can be observed. Figure 21b,d,f show the stress distribution and deformation behaviors of these models before reaching the ultimate state. From these figures, it can be concluded that the yielding and collapse of both the plate and stiffeners occur almost simultaneously during the lateral loading process of such stiffened plates. As shown in Figure 16a–c, the buckling of plates is quite significant at the ultimate state, and the deformations of stiffeners has also become more severe.



Figure 18. The collapse modes and stress distribution at the ultimate state of models 'R1S1': (a) Model 'R1T1S1'; (b) Model 'R1T2S1'; (c) Model 'R1T3S1'; (d) Model 'R1T4S1'. Scale: 10.







Figure 20. Cont.



Figure 20. The front views of the collapse modes of model 'R3S1': (**a**) Model 'R3T1S1'; (**b**) Model 'R3T2S1'; (**c**) Model 'R3T3S1'; (**d**) Model 'R3T4S1'. Scale: 15.



Figure 21. The collapse modes and stress distributions of model 'R5', 'R6', and 'R8': (**a**) ultimate state, model 'R5T1S1'; (**b**) before the ultimate state, model 'R5T1S1' (**c**) ultimate state, model 'R6T1S1'; (**d**) before the ultimate state, model 'R6T1S1'; (**e**) ultimate state, model 'R8T1S1'; (**f**) ultimate state, model 'R8T1S1'. Scale: 10.

4. Empirical Formula of the Lateral Ultimate Strengths

Although the NFEM codes is a powerful tool for structural strength analysis, it requires a relatively long time to calculate and process data. Therefore, in practical structure design, it is highly desirable to use an empirical formula that contains key variables and provides a straightforward prediction for the lateral ultimate strength of stiffened plates. Before the empirical formula is presented, certain preparations have been made. First of all, the design variables have been confirmed based on experiences from structural design and analysis. Next, the variables for the formula are selected. Then, an appropriate form of the formula is constructed, considering the analyzing examples and reflecting the impact of geometrical properties on ultimate strengths. Finally, a formula is derived using the minimum mean-square error method and the NFEM results.

There are various empirical formulae for predicting the axial ultimate strength of the stiffened plates, which can provide a reasonable reference for the choice of the structure of the empirical formula for the lateral ultimate strength of the doubly curved stiffened plates. Some are only related to plate slenderness (Faulkner [23] and Paik [24]), Moreover, there are two influencing factors in other empirical formulae, including the plate slenderness and the column slenderness. As mentioned in the analysis above, both the plate's slenderness ratio and the column's slenderness ratio are the key influence factors of the lateral ultimate strength of the doubly curved stiffened plates.

Zhang and Khan [21] presented an empirical formula for calculating the axial compression ultimate strength of a stiffened plate, as shown in Equation (6). Based on this form, Shi [25] derived an empirical formula for evaluating the axial strength of the U-type stiffened plates, as shown in Equation (7). From the formula, we can discover that there are three coefficients selected according to the specific situation. The coefficients in such a form can signify certain characteristic of different models and load cases and it is well worth applying in other situations.

$$\frac{\sigma_{UA}}{\sigma_Y} = \frac{1}{\beta^{0.28}} \frac{1}{\left(1 + \lambda^{3.2}\right)^{0.5}}$$
(6)

$$\frac{\sigma_{UA}}{\sigma_Y} = \frac{1}{\beta^{0.2}} \frac{1}{(1+\lambda^{20})^{0.05}}$$
(7)

Taking the variations in the curvatures and sections of the stiffeners into consideration, a series of doubly curved plates models are established, and the lateral ultimate strength can be calculated by employing the NFEM and the results are shown in the Appendix. The influence of the curvatures on the ultimate strength is one of the main concerns. Based on these results, using the basis function fitting method and the minimum mean-square error (MMSE) technique, the corresponding ultimate strength prediction formulae in terms of slenderness ratios λ and β and the curvature angles θ_L and θ_T are derived.

Based on the form of Equation (6), the empirical formula for predicting the lateral ultimate strength of the doubly curved stiffened is expressed as follows. To consider the effect of the curvature in longitudinal and transversal directions, the coefficients are replaced by a series of rational low-order polynomial functions. The basic form of the empirical formula is shown in Equation (8), and functions α_1 , α_2 , and α_3 are expressed in Equations (9)–(11), where the parameters α_1 , α_2 , and α_3 denote the influences of the curvature.

$$\frac{\sigma_U}{\sigma_Y} = \frac{1}{\beta^{\alpha_1}} \frac{1}{\left(1 + \lambda^{\alpha_2}\right)^{\alpha_3}} \tag{8}$$

$$\alpha_1 = 2.489 - 61.76\theta_T - 24.98\theta_L + 345.4(\theta_T)^2 + 771\theta_L\theta_T + 78.29(\theta_L)^2 -2398(\theta_T)^2\theta_L - 1549\theta_T(\theta_L)^2 - 71.21(\theta_L)^3$$
(9)

$$\alpha_{2} = 0.4812 - 0.02773\theta_{T} - 3.222\theta_{L} - 26.46(\theta_{T})^{2} - 95.56\theta_{L}\theta_{T} + 21.94(\theta_{L})^{2} - 959.5(\theta_{T})^{2}\theta_{L} - 13.93\theta_{T}(\theta_{L})^{2} - 33.28(\theta_{L})^{3}$$
(10)

$$\alpha_{3} = 7.671 + 44.03\theta_{T} - 2.319\theta_{L} - 469.6(\theta_{T})^{2} + 295.2\theta_{L}\theta_{T} + 94.74(\theta_{L})^{2} + 2716(\theta_{T})^{2}\theta_{L} - 2848\theta_{T}(\theta_{L})^{2} - 170.4(\theta_{L})^{3}$$

$$(11)$$

Comparisons with results derived from the formula and the practical NFEM are shown in Figure 22. We can find that the prediction of the formula and the NFEM results match well, with $R^2 > 0.95$. We can also find that the lateral ultimate strengths of the doubly curved stiffened plates decrease with the increase of the parameters λ and β from the picture; moreover, the difference distribution frequency is shown in Figure 23.

A stiffened plate captured from an oil tanker ($\theta_L = 0.022$ and $\theta_T = 0.13$) is calculated using the formula and the NFEM, and the comparison between the two methods is shown in Figure 24. The values obtained from both methods are similar, and the difference between them is around 5%. And the results from the formula are slightly smaller than those from the NFEM.



Figure 22. Cont.



Figure 22. Formulae predictions based on the NFEM results: (a) Model 'R1' $R^2 = 0.9894$; (b) Model 'R2', $R^2 = 0.952$; (c) T Model 'R3', $R^2 = 0.9679$; (d) Model 'R4', $R^2 = 0.9631$.



Figure 23. Difference distribution frequency.



Figure 24. Comparison between the formula and the NFEM.

5. Conclusions

In this paper, the collapse modes and ultimate strengths of doubly curved stiffened plates under lateral pressure have been investigated by using the NFEM, where different groups of curvature angles have been considered and the influences of plates and stiffeners scantlings have been studied. Meanwhile, the deformation behaviors and collapse sequence of target models in lateral loading cases have been also studied. The following conclusions can be drawn:

- (1) The curvature angle, along the direction of stiffeners, has significant impact on the lateral ultimate strength of the doubly curved stiffened plate. The normal bearing capacity of the stiffener enhances as the curvature of stiffeners increases. As a result, a stronger constraint is applied to the plates, which strengthens the lateral bearing capacity and the ultimate strength of the doubly curved stiffeners.
- (2) In the lateral loading process, yielding firstly happens in the areas around the endings of the stiffeners. In most load cases, the stiffeners collapse before the plates. In actual designs, enhancing the strength of the stiffeners is a more effective way to improve the lateral loading capacity.
- (3) The collapse modes of the doubly curved stiffened plates can be concluded as the combination of global buckling and the local buckling of the plates. If the strength of the stiffener and the plates are similar or if the column slenderness is quite large, the global buckling will become the dominant collapse mode. The local buckling of the plates is significant in the cases with strong stiffeners and weak plates.
- (4) A new empirical formula Equation (4) is proposed for lateral ultimate strength prediction of doubly curved stiffeners. The proposed formula is applicable with $R^2 > 0.95$ comparing with 144 NFEM results. And the average absolute differences between the results from the formula and the NFEM is 3.7%. Over the wide range of λ (0–1.5) and β (0.6–3) considered in this paper, the agreement between the proposed formula and numerical results is very good. Additionally, the differences between the values from the formula and the NFEM on a practical case are around 5%, which can be acceptable in engineering.

Author Contributions: Conceptualization, J.C.; methodology, J.C.; software, G.G.; validation, G.G.; writing—review and editing, G.G. and J.C.; supervision, D.W.; project administration, J.C. and D.W.; funding acquisition, J.C. and D.W. All authors have read and agreed to the published version of the manuscript.

Funding: This research was funded by NSFC (No. 52371328, No. 51809167, No. 51979163 and No. U2241266) and the Fundamental Research Funds for the Central Universities.

Data Availability Statement: Data is contained within the article.

Conflicts of Interest: The authors declare no conflict of interest.

Nomenclature

Notations	Descriptions
а	Plate length
b	Plate breath
t _p	Thickness of attached plate
t_w	Web thickness of the stiffener
h_w	Web height of the stiffener
t _f	Flange thickness of the stiffener
\dot{b}_f	Flange breadth of the stiffener
u _C	The magnitude displacement of the doubly curved stiffened plate's central point
Ε	Elastic modulus
Ι	The moment of inertia of the stiffener with the attached plate
А	Area of transverse section of the stiffened plate
r	Gyration radius of the stiffened plate
σ_Y	Yield stress of the material
σ_U	The normalized lateral ultimate stress of the ultimate state (Defined as the ratio of the lateral ultimate load corruing capacity to the sectional area and the value of the
	vield stress.)
(Track	The normalized axial ultimate stress of the ultimate state (Defined as the ratio of
υUA	the axial ultimate load-carrying capacity to the sectional area and the value of the
	yield stress.)
λ	Column slenderness
β	Plate slenderness
R_L	Longitudinal curve radius of the plate
R_T	Transversal curve radius of the plate
$ heta_L$	Longitudinal curvature angle
θ_T	Transversal curvature angle
$\alpha_i (i = 1, 2, 3)$	Parameters in terms of curvature angles

Appendix A. Structural Dimensions, Properties, and NFEM Solutions of the Models

R_L	R_T	а	b	θ_L	θ_T	t_p	h_w	t_w	b_f	t_f	σ_Y	Ε	λ	β	σ_U/σ_Y
mm	mm	mm	mm	-	-	mm	mm	mm	mm	mm	MPa	MPa	-	-	-
						24	360	20	90	20	355	206,000	0.368	0.605	0.00568
						14	360	20	90	20	355	206,000	0.366	1.038	0.00506
						11	360	20	90	20	355	206,000	0.369	1.321	0.00466
						8	360	20	90	20	355	206,000	0.375	1.816	0.00431
						24	330	20	80	18	355	206,000	0.41	0.605	0.00504
						14	330	20	80	18	355	206,000	0.404	1.038	0.00431
						11	330	20	80	18	355	206,000	0.406	1.321	0.004
14 522	40.306	4000	250	0.00979	0.275	8	330	20	80	18	355	206,000	0.412	1.816	0.00361
14,525	40,300	4000	350	0.00868	0.275	24	280	20	60	12	355	206,000	0.518	0.605	0.00365
						14	280	20	60	12	355	206,000	0.503	1.038	0.003
						11	280	20	60	12	355	206,000	0.501	1.321	0.00284
						8	280	20	60	12	355	206,000	0.504	1.816	0.00250
						24	220	20	50	10	355	206,000	0.684	0.605	0.00254
						14	220	20	50	10	355	206,000	0.652	1.038	0.00197
						11	220	20	50	10	355	206,000	0.646	1.321	0.00195
						8	220	20	50	10	355	206,000	0.644	1.816	0.00175

R_L	R_T	а	b	θ_L	θ_T	t_p	h_w	t_w	b_f	t_f	σ_Y	Е	λ	β	σ_U/σ_Y
mm	mm	mm	mm	-	-	mm	mm	mm	mm	mm	MPa	MPa	-	-	-
						24	360	20	90	20	355	206.000	0.368	0.605	0.00575
						14	360	20	90	20	355	206,000	0.366	1.038	0.00496
						11	360	20	90	20	355	206,000	0.369	1.321	0.00466
						8	360	20	90	20	355	206,000	0.375	1.816	0.00425
						24	330	20	80	18	355	206,000	0.41	0.605	0.00499
					0.234	14	330	20	80	18	355	206,000	0.404	1.038	0.00439
						11	330	20	80	18	355	206,000	0.406	1.321	0.00397
17,075 48,	48,849	4000	350	0.00717		8	330	20	80	18	355	206,000	0.412	1.816	0.00361
	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	1000	000	01007 17	0.201	24	280	20	60	12	355	206,000	0.518	0.605	0.0038
						14	280	20	60	12	355	206,000	0.503	1.038	0.00337
						11	280	20	60	12	355	206,000	0.501	1.321	0.00314
						0 24	280	20	60 50	12	355	206,000	0.504	1.616	0.00270
						14	220	20	50	10	355	200,000	0.652	1.038	0.00237
						11	220	20	50	10	355	206,000	0.646	1.321	0.00195
						8	220	20	50	10	355	206,000	0.644	1.816	0.00169
						24	360	20	90	20	355	206,000	0.218	0.605	0.0148
						14	360	20	90	20	355	206,000	0.217	1.038	0.0118
						11	360	20	90	20	355	206,000	0.219	1.321	0.0113
5719 45,						8	360	20	90	20	355	206,000	0.222	1.816	0.0105
						24	280	20	60	16	355	206,000	0.298	0.605	0.0109
						14	280	20	60	16	355	206,000	0.29	1.038	0.00831
						11 °	280	20	60	16 16	355	206,000	0.29	1.321	0.00749
	45,560	2375	350	0.00768	0.415	0 24	260	20	60 40	10	355	206,000	0.292	1.616	0.00690
						14	110	20	40	10	355	200,000	0.78	1.038	0.00313
						11	110	20	40	10	355	206,000	0.76	$\begin{array}{cccc} 1.038 & 0.00831 \\ 1.321 & 0.00749 \\ 1.816 & 0.00690 \\ 0.605 & 0.00515 \\ 1.038 & 0.00352 \\ 1.321 & 0.00298 \\ 1.816 & 0.00230 \\ 0.605 & 0.00462 \\ 1.038 & 0.00306 \\ 1.321 & 0.00256 \\ 1.816 & 0.00195 \\ \end{array}$	
						8	110	20	40	10	355	206,000	0.74	1.816	0.00230
						24	80	20	40	10	355	206,000	1.138	0.605	0.00462
						14	80	20	40	10	355	206,000	1.061	1.038	0.00306
						11	80	20	40	10	355	206,000	1.028	1.321	0.00256
						8	80	20	40	10	355	206,000	0.993	1.816	- 0.00575 0.00496 0.00466 0.00425 0.00499 0.00397 0.00397 0.00314 0.00270 0.00237 0.002 0.00195 0.00195 0.00195 0.00169 0.0148 0.0113 0.0105 0.0109 0.00831 0.00749 0.00690 0.00515 0.00298 0.00230 0.00256 0.00298 0.00256 0.00256 0.00151 0.00256 0.00151 0.0120 0.0112 0.00151 0.00731 0.00731 0.00552 0.00731 0.00552 0.00372 0.00372 0.00314 0.00789 0.00731 0.00552 0.00372 0.00314 0.00552 0.0036 0.00731 0.00552 0.00314 0.00256 0.00270 0.00314 0.00252 0.00314 0.00252 0.00314 0.00270 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00450 0.00455 0.00314 0.00195 0.00450 0.00450 0.00455 0.00294 0.00550 0.00294 0.00156
						24	360	20	90	20	355	206,000	0.218	0.605	0.0151
						14	360	20	90	20	355	206,000	0.217	1.038	0.0120
						0	360	20	90	20	355	206,000	0.219	1.321	0.0112
						24	280	20	90 60	20 16	355	200,000	0.222	0.605	0.01115
						14	280	20	60	16	355	206,000	0.29	1.038	0.00869
						11	280	20	60	16	355	206,000	0.29	1.321	0.00789
	- 4 0 - 2 0			0.00/10		8	280	20	60	16	355	206,000	0.29	1.816	0.00731
5475	54,020	2375	350	0.00648	0.434	24	110	20	40	10	355	206,000	0.84	0.605	0.00552
						14	110	20	40	10	355	206,000	0.78	1.038	0.00372
						11	110	20	40	10	355	206,000	0.76	1.321	0.00314
						8	110	20	40	10	355	206,000	0.74	1.816	0.0024
						24	80	20	40	10	355	206,000	1.14	0.605	0.00522
						14	80	20	40	10	355	206,000	1.061	1.038	0.00326
						11 °	80	20	40	10	355	206,000	1.028	1.321	0.00270
						20	200	11	100	20	355	200,000	0.995	1.010	0.00207
						20 24	280 280	11 11	120 120	20 20	355 355	206,000 206,000	0.284	1.349	0.00450
						16	280	11	120	20	355	206.000	0.268	1.69	0.00316
						10	280	11	120	20	355	206,000	0.25	2.7	0.002
						20	320	11	60	15	355	206,000	0.305	1.349	0.00432
						24	320	11	60	15	355	206,000	0.319	1.12	0.00613
						16	320	11	60	15	355	206,000	0.29	1.69	0.00314
74,243	4770	2400	650	0 136	0 0222	10	320	11	60	15	355	206,000	0.267	2.7	0.00195
11/210	T//U	2 1 00	000	0.130	0.0525	20	210	11	60	15	355	206,000	0.475	1.349	0.00425
						24	210	11	60	15	355	206,000	0.499	1.12	0.00550
						16	210	11	60	15	355	206,000	0.448	1.69	0.00294
						10	∠10 120	11 11	6U 40	15	305	206,000	0.404	2.7	0.00156
						20	120	11	40	10	333	200,000	1.01	1.349	0.00354

206,000 206,000

206,000

206,000

1.064

0.958

0.842

1.12

1.69

2.7

0.00456

0.00255

0.00104

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R _L	R_T	а	b	θ_L	θ_T	t_p	h_w	t_w	b_f	t_f	σ_Y	Е	λ	β	σ_U/σ_Y
mm	mm	mm	mm	-	-	mm	mm	mm	mm	mm	MPa	MPa	-	-	-
						20	320	12	120	20	355	206,000	0.32	1.401	0.00417
						16	320	12	120	20	355	206,000	0.308	1.751	0.00337
						12	320	12	120	20	355 355	206,000	0.296	2.335	0.00230
						20	280	12	120	20	355	206,000	0.363	1.401	0.00470
					0.232	16	280	12	120	20	355	206,000	0.349	1.751	0.00327
						12	280	12	120	20	355	206,000	0.335	2.335	0.00224
13,104	7198	3044	675	0.0938		22	280	12	120	20	355	206,000	0.37	1.274	0.00457
						20	220	16 16	60 60	16 16	355	206,000	0.567	1.401	0.00366
						10	220	16	60	16	355	206,000	0.503	2.335	0.00271
						22	220	16	60	16	355	206,000	0.581	1.274	0.00447
						20	180	16	40	12	355	206,000	0.764	1.401	0.00335
						16	180	16	40	12	355	206,000	0.724	1.751	0.00253
						12	180	16 16	40 40	12	355	206,000	0.679	2.335	0.00167
						22	100	10	40	12	355	200,000	0.782	1.2/4	0.00575
						24	360	20	90	20	355	206,000	0.218	0.605	0.0124
						12	360	20	90 90	20	355	206,000	0.218	1.21	0.0112
						6	360	20	90	20	355	206,000	0.227	2.422	0.0100
						24	200	20	80	10	355	206,000	0.424	0.605	0.00584
				0.00588		12	200	20	80	10	355	206,000	0.402	1.21	0.00472
11,500 59						10	200	20	80	10	355	206,000	0.4	1.453	0.00433
	59,500	2375	350		0.207	6 24	200	20	80 80	10	355	206,000	0.403	2.422	0.00373
						24 12	140	20	80 80	10	355	206,000	0.557	1.21	0.00369
						10	140	20	80	10	355	206,000	0.551	1.453	0.00280
						6	140	20	80	10	355	206,000	0.548	2.422	0.00222
						24	100	20	80	10	355	206,000	0.818	8 2.422 0.00222 .8 0.605 0.00273 .6 1.21 0.00191 .4 1.453 0.00180	
						12	100	20	80	10	355	206,000 0.756 1.21 0 206,000 0.744 1.453 0	0.00191		
						10	100	20	80 80	10	355	206,000	0.744	1.453	0.00180
						24	100	10	120	10	255	200,000	0.75	1.20	0.00205
						24 20	420 420	10	120	16	355	206,000	0.296	1.58	0.00303
						18	420	10	120	16	355	206,000	0.200	1.84	0.00263
						14	420	10	120	16	355	206,000	0.263	2.37	0.00222
						24	320	10	120	16	355	206,000	0.391	1.38	0.00249
						20	320	10	120	16	355	206,000	0.373	1.66	0.00230
						18 14	320	10	120	16 16	355	206,000	0.363	1.84 2.37	0.00217
26,500	37,500	3200	800	0.0213	0.121	24	280	10	80	10	355	206,000	0.544	1.38	0.00100
						20	280	10	80	12	355	206,000	0.501	1.66	0.00180
						18	280	10	80	12	355	206,000	0.486	1.84	0.00174
						14	280	10	80	12	355	206,000	0.455	2.37	0.00145
						24	190	10	80 80	6	355	206,000	0.933	1.38	0.00146
						18	190	10	80	6	355	206,000	0.855	1.84	0.00135
						14	190	10	80	6	355	206,000	0.793	2.37	0.00114
						24	420	10	120	16	355	20,600	0.296	1.38	0.00303
						20	420	10	120	16	355	20,600	0.283	1.66	0.00270
						18	420	10	120	16	355	20,600	0.277	1.84	0.00248
						14 24	420 320	10 10	120 120	16 14	355	20,600	0.263	2.37	0.00221
						24 20	320 320	10	120	10 16	355	206.000	0.373	1.50	0.00244
						18	320	10	120	16	355	206,000	0.363	1.84	0.00205
41 600	27 200	2200	800	0.0204	0.0740	14	320	10	120	16	355	206,000	0.344	2.37	0.00175
T1,000	21,200	5200	000	0.0294	0.0769	24	280	10	80	12	355	206,000	0.529	1.38	0.00203
						20	280	10	80	12	355	206,000	0.501	1.66	0.00174
						18 14	280	10 10	80 80	12 12	355	206,000	0.486	1.84 2.27	0.00167
						14 24	∠o0 190	10	80 80	12 6	355	206.000	0.435	2.37 1.38	0.00136
						20	190	10	80	6	355	206,000	0.883	1.66	0.00130
						18	190	10	80	6	355	206,000	0.855	1.84	0.00121
						14	190	10	80	6	355	206,000	0.793	2.37	0.00112

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