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Abstract: We numerically investigate the supratransmission phenomenon in an active nonlinear system modeled by the 1D/2D discrete sine-Gordon equation with non-local feedback. While, at a given frequency, the typical passive system exhibits a single amplitude threshold marking the onset of the phenomenon, we show that the inclusion of non-local feedback manifests additional thresholds that depend upon the specific boundary from which supratransmission is stimulated, realizing asymmetric (i.e., non-reciprocal) dynamics. The results illustrate a new means of controlling nonlinear wave propagation and energy transport for, e.g., signal amplification and mechanical logic.

Keywords: supratransmission; non-reciprocity; active mechanical lattices; sine-Gordon equation

1. Introduction

In the context of elastodynamics, phononic crystals and metamaterials [1,2]—collectively phononic materials—are two classes of materials whose artificial microstructure provides for the management of mechanical waves. For linear, small-amplitude waves, the microstructure design regulates internal scattering and resonance phenomena such that the Fourier components of a disturbance penetrate the material within only particular frequency ranges and in all or specific directions; outside these pass bands, i.e., within the band gaps, wave propagation is prohibited, the associated wave energy decaying exponentially in space. This filtering capability has inspired proposals for a variety of phononic material applications [3–5]. Nevertheless, nonlinearities inherent to the microstructure enable a unique dynamic response for large amplitude waves. Supratransmission describes the spontaneous flow of energy within the band gap via the nonlinear (non-topological) modes of a medium activated by a boundary driving of sufficient amplitude [6]. The effect is a generic property of nonlinear systems, having been shown to emerge from both integrable and non-integrable governing equations [7–12] even when accounting for dissipative effects [13] which more readily extinguish their linear counterparts. This may be exploited, e.g., for the transmission of binary, nonlinear signals in lightly-damped systems [14], as well as for the digital amplification of exceptionally weak signals for sensing [9,12,15,16]. While these and other studies have promoted an understanding of amplitude-dependent energy transmission, investigations of systems with a directional response, beneficial in applications for greater control of energy flow, are few.

Reciprocity describes the symmetry of wave transmission between two points in space: if a source and receiver exchange positions, the corresponding frequency response function is identical even in the presence of inhomogeneities and losses [17]. In recent years, inspired by the concept of electric [18] and optical [19,20] diodes and motivated by their application in communications, sensing, and the directional control of energy flow, significant effort has been directed toward the discovery and study of non-reciprocity in other domains of physics. Regarding phononic materials [3,21–24], inherent non-reciprocity has been demonstrated in a number of systems utilizing unique and intersecting strategies for microstructure design, i.e., internal architectures characterized by internal



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motion [25,26], time-dependent [27–29] and topological [30–32] properties, and nonlinearity [33–35]. The focus of these and parallel studies is linear wave manipulation. Transition waves—nonlinear (topological) modes characteristic of multi-stable systems which propagate by liberating stored elastic energy—demonstrate non-reciprocity as well [36,37]; however, they are energetically limited to one-time operation or short propagation distances [38]. In the context of supratransmission, Wu et al. [39] exploited a spatial asymmetry in the metamaterial construction to elicit non-reciprocal transmission.

While most of the previous references employ a passive material platform, a small but growing collection [25,28,29,31,32,34] investigate linear wave propagation within in an active setting. Recently, feedback control integrated directly into the material architecture has opened the door to more complex interactions unavailable in traditional structures and have been shown to be capable of eliciting non-reciprocal behavior [32,40]. Uniquely, rather than injecting energy into (or extracting from) the system by an external means, feedback involves observing the state of the system and then, following predetermined relations, generating a response that alters the present state. As a result, the material behavior is inherent rather than a function of environmental conditions (e.g., temperature, external fields, pumps, actuators which behave independently of the system). At present, the supratransmission phenomenon in feedback mediated nonlinear networks has yet to be investigated.

In this article, we present an approach to asymmetric (i.e., non-reciprocal) wave propagation in nonlinear mechanical networks with a focus on energy transmission within the band gap. Unique in the supratransmission literature, these lattice materials incorporate active elements which impart a local, non-conservative forcing proportional to non-local degrees-of-freedom. The result of this construction is that the onset of the supratransmission phenomenon is not only a function of the driving parameters (i.e., frequency and amplitude) but the specific network boundary at which the excitation is applied and from which wave energy is transmitted to the medium, establishing the asymmetric dynamic behavior.

The article is organized as follows: Section 2 presents the nonlinear governing equation of a representative, one-dimensional mechanical system with feedback and formulates the corresponding dispersion relation of linear, small-amplitude dynamics. In Section 3, we analyze the supratransmission characteristics of the system, demonstrating asymmetric performance. We also present results for a two-dimensional network. Section 4 concludes the article with a summary of the main results and proposals for future research directions.

2. Theory

2.1. Model

To demonstrate the feedback-mediated asymmetric energy transmission, we initially consider the nonlinear dynamics of a one-dimensional (1D) periodic network of coupled pendula, a modified version of that analyzed by Geniet and Leon [6] (Figure 1a). The pendulum motif is composed of a ring of radius, ℓ , and mass, *m*, concentrated at a single point along the circumference. The pendulum rotates (in-plane), φ , about its center which, in the presence a gravitational field of strength g, adjusts the local potential as $\psi = mg\ell(1 - \cos \varphi)$, which describes a non-convex energy landscape responsible for the network nonlinearity (Figure 1b). Moreover, the potential renders the system multi-stable, possessing several energetically degenerate ground states at $\varphi = 2\pi p$, $p \in \mathbb{Z}$. Utilizing elastic bands to manifest an effective (torsional) stiffness, $c = k\ell^2$, the network emerges via nearest-neighbor coupling. Uniquely, a feedback mechanism imposes an additional, non-conservative influence on each φ_i which, from myriad possible descriptions, we relate to non-local variables; specifically, the torque, f_c , applied to φ_i via feedback is proportional to the relative rotation of its nearest neighbors, i.e., $f_c = s(\varphi_{j+1} - \varphi_{j-1})$, where *s* denotes the proportional control gain. Physically, the feedback mechanism may emerge from active components which sense the non-local displacement and then, through a connected microcontroller and actuator, apply the calculated torque. For example, each pendulum may

connect to a potentiometer which converts the angular displacement to a predetermined voltage drop measured by a programmable micro-controller. Dependent upon the voltage associated with φ_{j+1} and φ_{j-1} , the micro-controller regulates a voltage sent to drive a DC motor which applies a torque, f_c , to the pendulum at site *j*. For an arbitrary pendulum in the 1D network, the non-dimensional governing equation has the form

$$\ddot{\varphi}_{j} + \bar{c}(2\varphi_{j} - \varphi_{j+1} - \varphi_{j-1}) + \bar{s}(\varphi_{j+1} - \varphi_{j-1}) + \bar{r}\sin\varphi_{j} = 0,$$
(1)

where $\bar{c} = c\tau^2/m$, $\bar{s} = s\tau^2/m\ell^2$, and $\bar{r} = g\tau^2/m\ell$ are dimensionless parameters. The period of oscillation for an isolated pendulum, $\tau = 2\pi\sqrt{\ell/g}$, is a natural choice for the characteristic time and the normalizing parameter defining the dimensionless temporal variable, $\bar{t} = t/\tau$. Absent the feedback mechanism (i.e., $\bar{s} = 0$), Equation (1) has a form that arises in several domains of science [41–43]; however, the sine-Gordon equation with feedback appears unique to the metamaterial described in this article.

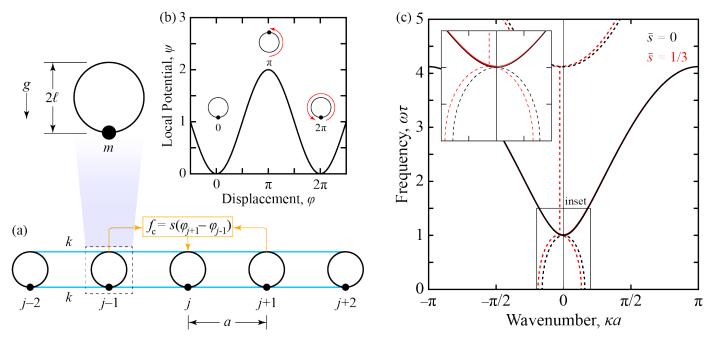


Figure 1. Pendulum network with feedback. (**a**) one-dimensional network of rotating pendula with elastic bands providing nearest-neighbor coupling and a feedback mechanism imposing a local influence which is proportional to the non-local displacements, $f_c = s(\varphi_{j+1} - \varphi_{j-1})$; (**b**) the on-site, multi-stable potential which manifests the system nonlinearity; (**c**) the complex dispersion of small-amplitude waves of prescribed non-dimensional frequency, $\omega \tau$, with real and imaginary wavenumbers, κ_R (solid) and κ_I (dotted), respectively. The feedback effect shifts the attenuating structure about $\kappa_I = 0$, provoking non-reciprocal wave propagation.

For a system of *N* pendula, Equation (1) forms the basis of a system of equations, $M\ddot{u} + (K + S)u + f_{NL} = 0$, where **M**, **K**, and **S**, respectively, are the mass, stiffness, and feedback matrices; **u** and f_{NL} are, accordingly, the displacement and nonlinear force vectors. While the mass and stiffness matrices are symmetric, $S \neq S^T$, which renders the system non-Hermitian and supportive of non-reciprocal dynamics. Alternatively, K + S may emerge from a system in which the spring interactions, when deformed, manifest unequal forces at each end. Moreover, from the continuum limit of Equation (1) with expansion $\varphi_{j\pm 1} \rightarrow \varphi \pm \varphi_{,\bar{x}} + \varphi_{,\bar{x}\bar{x}} / 2$,

$$\varphi_{,\bar{t}\bar{t}} - \bar{c}\varphi_{,\bar{x}\bar{x}} + 2\bar{s}\varphi_{,\bar{x}} + \bar{r}\sin\varphi_{j} = 0,$$

where $\bar{x} = x/a$, it is apparent that the activity either injects or extracts energy (spatially) from the system depending on the sign of $\bar{s}\varphi_{rx}$. Consequently, for a positive \bar{s} and, e.g., an exponentially decreasing (increasing) displacement function, the active term tends to support (dampen) the motion of φ . Odd spatial derivatives of higher order would have a similar effect.

2.2. Dispersion

The proposed system supports a number of amplitude-dependent solutions owing to the nonlinear on-site potential. For the particular case of small-amplitude motion, the Fourier components of an initially compact disturbance disperse as the propagation speeds depend on the corresponding wavelength. To establish the dispersion relation, we first linearize Equation (1) about a (stable) equilibrium configuration and then theoretically extend the pendulum network to infinity through the application of Bloch boundary conditions on a single unit cell; thus, the dynamics of such a system are described by $\varphi_j = \Phi e^{i(\kappa j a - \omega \tau I)}$ with κ and ω , respectively, the spatial and temporal frequencies. Applied to Equation (1), the Bloch wave solution entails the following dispersion relation:

$$-\omega^{2}\tau^{2} + \bar{c}(2 - \gamma - 1/\gamma) + \bar{s}(\gamma - 1/\gamma) + \bar{r} = 0,$$
(2)

where $\gamma = e^{i\kappa a}$. The complex band structure emerges from the solution, $\gamma(\omega\tau)$, which contains the real and imaginary components of the normalized wavenumber, $\kappa a = \kappa_R a + i\kappa_I a$: $\kappa_R a = \text{Re}[-i\ln(\gamma)]$, specifying the spatial oscillation of displacement, and $\kappa_I a = \text{Im}[-i\ln(\gamma)]$, expressing the spatial attenuation of the amplitude.

3. Results

3.1. Simulation

In the following, we numerically investigate the dynamics of a representative nonlinear network with non-local feedback. To this end, we consider a finite, nonlinear system of *N* pendula subject to a prescribed harmonic boundary displacement, $\varphi_j = \Phi \sin(\omega \tau \bar{t})$, where $\omega \tau$ is set within the band gap and Φ varies between simulations. The left (L) and right (R) boundaries are distinguished, respectively, by j = 1 and j = N. The system response is quantified by the mean energy transmitted to the network by the driven boundary over an $n \in \mathbb{Z}^+$ multiple of the excitation period, *T*:

$$E_{\rm in} = \frac{\bar{c}}{nT} \int_0^{nT} (\varphi_2 - \varphi_1) \dot{\varphi}_1 d\bar{t},$$
 (3)

for the left boundary; for the right boundary, substitute the indices in Equation (3) as $\{1,2\} \rightarrow \{N, N-1\}$. For sufficiently small amplitudes, the system exhibits a linear response where the energy supplied to the system, ultimately, returns to the driving due to Bragg reflection, causing E_{in} to vanish. Conversely, beyond a critical amplitude, Φ_c , the driving excites nonlinear modes which penetrate the system, resulting in $E_{in} > 0$, signifying a spontaneous energy flow. While a number of methods have been developed to predict the amplitude threshold [7,44–47], in this article, we intend to introduce the concept of active lattices to the supratransmission literature and initiate exploration of its effects.

Simulations evolve Equation (1) for a network of N = 1000 coupled pendula with $\bar{c} = 4$ and $\bar{r} = 1$ for n = 100 periods. In order to minimize reflections from the free boundary (thus, mimicking an infinite medium), we apply a linearly increasing viscous damping, $\bar{\eta} \dot{\phi}_j$, to the final 800 sites with max $(\bar{\eta}) = 1/10$. In order to avoid the shock wave generated by vanishing initial velocities, the simulations adopt an inaugural velocity profile matching the (linear) evanescent solution at the driving frequency, i.e., $\dot{\phi}_j = -\omega\tau\Phi \cos[(j-1)\kappa_{\rm R}a]e^{-(j-1)\kappa_{\rm I}a}$. In simulation, each boundary is excited in turn, revealing a direction-specific energy transmission for $\bar{s} \neq 0$.

3.2. Supratransmission: One Dimension

Figure 1c graphs the dispersion diagram for instances of passive ($\bar{s} = 0$) and active $(\bar{s} = 1/3)$ feedback control, each exhibiting a single pass band separating two band gaps. Apparently, for the reference case where $\bar{s} = 0$, the diagram is symmetric about $\kappa = 0$, implying that wave propagation is independent of direction. Harmonic boundary driving in the pass band transmits wave energy unabated into the network; in the band gap, Bragg reflection confines energy to the boundary as indicated by $\kappa_{\rm I} \neq 0$. Activating the feedback mechanism modifies the dispersion. While the real component of the band structure, $\kappa_{\rm R}(\omega\tau)$, remains symmetric about $\kappa = 0$, the imaginary component, $\kappa_{\rm I}(\omega\tau)$, shifts primarily along the wavenumber axis, breaking the diagram symmetry. This implies that wave propagation, including the supratransmission phenomenon, is direction-dependent under non-local feedback control. This is because the onset of supratransmission is linked to the critical amplitude which destabilizes the evanescent wave profile [48], whose feedback has not been made direction-dependent. Analyzing small-amplitude waves, Rosa and Ruzzene [32] also observed non-reciprocal behavior via a feedback effect that was attributed to a complex temporal frequency whose imaginary component resulted in a propagation direction dependent exponential growth or decay. This article describes a similar effect for nonlinear waves which is attributed to a shift in the imaginary wavenumber component of the linear wave dispersion.

Figure 2a plots the energy transmission efficiency of a harmonic boundary driving within the lower band gap ($\omega \tau = 0.9$) in the pendulum network as a function of the driving amplitude for two values of the control gain, $\bar{s} = 0$ and $\bar{s} = 1/3$. In general, below a critical amplitude, Φ_c , the linear, small-amplitude response of the system dominates as the total energy remains concentrated near the driven boundary. As the driving amplitude increases, however, higher harmonic modes—the multiples of the driving frequency which appear in the pass band—generated by burgeoning nonlinear effects propagate into the system, contributing to a relatively small increase in E_{in} . For the reference case ($\bar{s} = 0$), beyond the critical amplitude predicted by Geniet and Leon [6], $\Phi_c = 1.77$, the attenuating displacement profile of the linear solution is unstable [48,49]; instead, nonlinear modes generated at the driven boundary subsequently propagate into the system, signified by the sudden increase in the transmitted energy, i.e., supratransmission. As the band diagram of the reference system is symmetric (Figure 1c), the critical amplitude for supratransmission is identical for left- and right-boundary driving. However, setting $\bar{s} \neq 0$ activates the feedback mechanism and adjusts the band diagram as described previously—for $\bar{s} = 1/3$ and $\omega \tau = 0.9$, forward and backward waves are attenuated according to $\kappa_{\rm I}a = 0.150$ and $\kappa_{Ia} = 0.317$, respectively—which suggests that Φ_{c} differs for the same excitation on opposite boundaries. The simulation results depicted in Figure 2a confirm that the amplitude thresholds for supratransmission in the feedback network do, indeed, differ: $\Phi_c^L = 1.41$ and $\Phi_c^R = 2.44$, respectively, for left- and right-boundary driving. Figure 2b shows how the two thresholds diverge as a function of \bar{s} .

The supratransmission demonstrated in the previous results transmits the energy of a signal with band-gap frequency through a nonlinear system. To further emphasize this phenomenon, Figure 2c displays snapshots of the left and right boundary regions of the active pendula network ($\bar{s} = 1/100$) subject to the same harmonic driving with various amplitudes. For visualization purposes, the instantaneous displacement in the boundary region is normalized such that max($|\varphi_i|$) = 1.

The superposed color indicates the corresponding energy distribution over the same region, $\mathcal{H}_j = \frac{1}{2}\dot{\varphi}_j^2 + \frac{1}{2}\bar{c}(\varphi_j - \varphi_{j-1})^2 + \psi(\varphi_j)$. As expected, for small amplitudes, the displacement profile exhibits an exponential decay and energy concentrates at the boundary. In the second row of Figure 2c, the excitation amplitude exceeds the supratransmission threshold for only the left boundary driving; consequently, the evanescent response is unstable and energy propagates away from the boundary. The same excitation amplitude does not exceed the critical value for the right boundary; therefore, apart from the aforementioned higher harmonics lying within the pass band, the energy remains localized there.

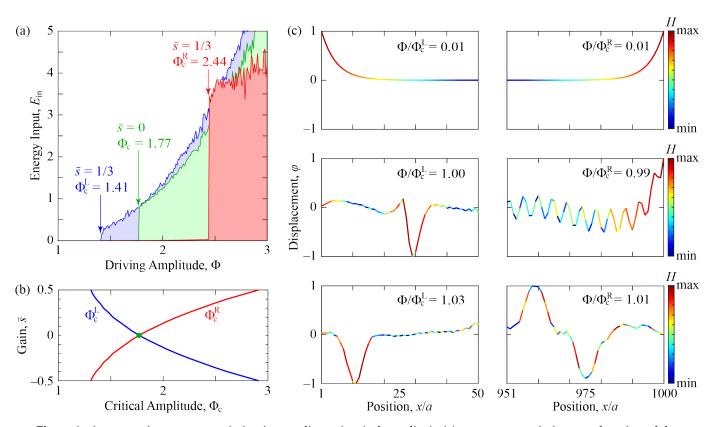


Figure 2. Asymmetric supratransmission in one dimension (color online). (a) energy transmission as a function of the driving amplitude. For supratransmission, the passive system ($\bar{s} = 0$) exhibits a common threshold amplitude, $\Phi_c = 1.77$, when excited from either the left or right boundary; conversely, the active system ($\bar{s} = 1/3$) expresses two thresholds, $\Phi_c^L = 1.41$ and $\Phi_c^R = 2.44$, when excited from the left and right, respectively; (b) the separation of Φ_c^L and Φ_c^R over a range of \bar{s} ; (c) the normalized displacement profile and energy distribution for an active system ($\bar{s} = 1/100$) at various amplitudes. For each row, the left and right boundaries are subject to the same harmonic excitation, the amplitude of which may be above/below the critical amplitude to trigger supratransmission at the particular boundary.

3.3. Supratransmission: Two Dimensions

The pendulum network can be extended to obtain a two-dimensional (2D) periodic system, e.g., a square lattice with nearest-neighbor coupling. Accordingly, the non-local feedback applied to each $\varphi_{j,k}$ depends on additional terms, $f_c = \bar{s}_1(\varphi_{j+1,k} - \varphi_{j-1,k}) + \bar{s}_2(\varphi_{j,k+1} - \varphi_{j,k-1})$ where \bar{s}_1 and \bar{s}_2 are gain parameters. Considering a Cartesian frame, indices *j* and *k* designate sites along the *x*- and *y*-axes, respectively, with the indices increasing with the relevant coordinate. Following the same normalization scheme as before, the governing equation for a generic unit cell is

$$\begin{aligned} \ddot{\varphi}_{j,k} &+ \bar{c} (4\varphi_{j,k} - \varphi_{j+1,k} - \varphi_{j-1,k} - \varphi_{j,k+1} - \varphi_{j,k-1}) \\ &+ \bar{s}_1 (\varphi_{j+1,k} - \varphi_{j-1,k}) + \bar{s}_2 (\varphi_{j,k+1} - \varphi_{j,k-1}) \\ &+ \bar{r} \sin \varphi_{j,k} = 0. \end{aligned}$$
(4)

Linearizing Equation (4) and applying the Bloch solution, $\varphi_{j,k} = \Phi e^{i(\kappa_x j a + \kappa_y k a - \omega \tau \bar{t})}$, yields the dispersion relation

$$-\omega^{2}\tau^{2} + \bar{c}(4 - \gamma_{x} - 1/\gamma_{x} - \gamma_{y} - 1/\gamma_{y}) + \bar{s}_{1}(\gamma_{x} - 1/\gamma_{x}) + \bar{s}_{2}(\gamma_{y} - 1/\gamma_{y}) + \bar{r} = 0,$$
(5)

where $\gamma_x = e^{i\kappa_x a}$ and $\gamma_y = e^{i\kappa_y a}$. Figure 3a plots the complex dispersion surfaces for the passive system ($\bar{s}_1 = \bar{s}_2 = 0$), revealing a single propagating surface separating two

attenuating surfaces within frequency band gaps. Since, as before, our focus is in the lower band gap where the real component of the wavevector vanishes, we substitute $\gamma_x = e^{-\kappa_{Lx}a}$ and $\gamma_y = e^{-\kappa_{Ly}a}$ in Equation (5) such that, for a given $\omega\tau$ in the lower band gap, the κ_{Lx} and κ_{Ly} which satisfy the relation trace the circular contour of the attenuating surface. Specifically, at the band-gap frequency $\omega\tau = 0.9$, the diagrams in Figure 3b compare the attenuating contours of various active systems with that of the passive lattice. Apparently, similar to the earlier 1D system, in the 2D network, activity causes the attenuating contour to shift relative to the passive reference centered at (κ_x, κ_y) = (0,0), indicating the broken symmetry of the system's small-amplitude dynamics which we expect to persist at larger amplitudes.

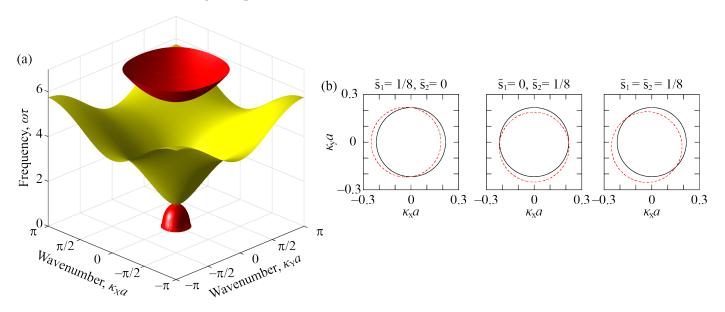


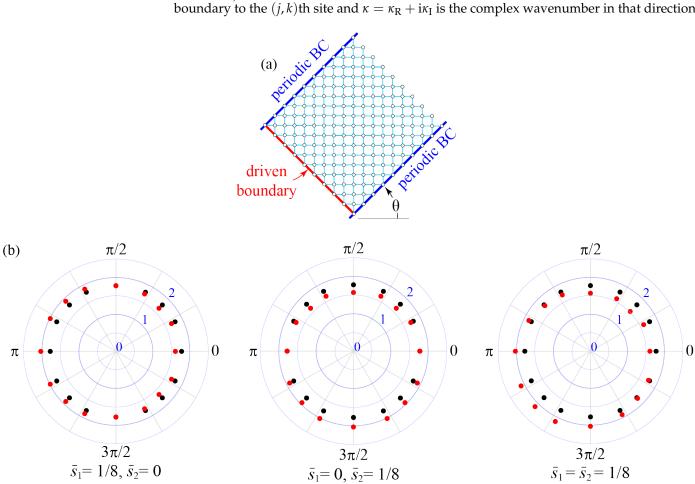
Figure 3. Dispersion (color online). (a) the complex dispersion surfaces for a passive, square lattice of pendula with near-neighbor interactions, including the propagating (yellow) and attenuating (red) modes; (b) the circular contour formed by the cross-section of the attenuating surface at band-gap frequency $\omega \tau = 0.9$. Active feedback, i.e., $\bar{s}_{1,2} \neq 0$, shifts the attenuation contour (red, dashed) with respect to the passive result (black), enabling direction-dependent non-reciprocal wave propagation.

To investigate asymmetric supratransmission in the 2D setting, we track the energy transmitted to a square lattice of finite dimension by a set of co-linear sites oscillating in phase at a band-gap frequency (Figure 4a). Beyond a critical amplitude, the driven boundary generates a wave front which propagates away from the driving and normal to it, i.e., at an angle θ with respect to the *x*-axis. Thus, the directional dependence of the supratransmission phenomena may be investigated by adjusting the slope of the driven boundary. To this end, we modify Equation (3) to accommodate the additional interactions of the 2D system under consideration:

$$E_{\rm in} = \frac{\bar{c}}{nT} \sum_{m,n} \sum_{r,s} \int_0^{nT} (\varphi_{r,s} - \varphi_{m,n}) \dot{\varphi}_{m,n} \mathrm{d}\bar{t},\tag{6}$$

where $\{m, n\}$ collects the indices of driven sites and $\{r, s\}$ the indices of their nearest neighbor(s). Thus, Equation (6) is the time-averaged energy transmitted to the lattice by the driven boundary.

Simulations utilize a square lattice with nearest-neighbor interactions to develop a rectangular system of approximately $23a \times 800a$ in dimension with the driven sites along one of the smaller dimensions and periodic boundary conditions along the two extended dimensions. For $\bar{c} = 4$ and $\bar{r} = 1$, the excitation frequency $\omega \tau = 0.9$ lies within the band gap. For n = 25 periods, we evolve the system following Equation (4) from an initial



velocity, $\dot{\phi}_{i,k} = -\omega \tau \Phi \cos(\kappa_R d) e^{\kappa_I d}$, where *d* is the perpendicular distance from the driven boundary to the (j, k)th site and $\kappa = \kappa_R + i\kappa_I$ is the complex wavenumber in that direction.

Figure 4. Asymmetric supratransmission in two dimensions (color online). (a) finite square lattice with periodic boundary conditions (blue) and a harmonically exciting line of nodes (red). (b) The threshold amplitude Φ_c versus orientation angle θ reveals the directional impact of activity for different combinations of the feedback parameters (red) relative to a passive system (black).

For a number of orientations, θ , we determine the critical driving amplitude above which E_{in} sharply increases. The panels in Figure 4b display the directional dependence of the threshold amplitude for different combinations of the feedback parameters, \bar{s}_1 and \bar{s}_2 . For the passive system, we determine a θ -independent threshold amplitude of $\Phi_{\rm c} = 1.77$ that is identical to that of the 1D passive system. However, following the wavenumber contours in Figure 3b, activity causes the threshold amplitude to shift relative to its passive value, the magnitude of the shift dependent upon θ . For the active scenario $(\bar{s}_1, \bar{s}_2) = (1/8, 0)$, the minimum amplitude for supratransmission decreases for waves propagating to the right (i.e., $|\theta| < \pi/2$) and increases for waves traveling to the left (i.e., $|\theta| > \pi/2$). The effect is maximized for $\theta = 0$ where the wave front aligns with the gradient, $\varphi_{,x}$, which determines the strength of f_c . Conversely, the threshold is unchanged at $\theta = \pm \pi/2$ where the effect of $\varphi_{,x}$ and, therefore, f_c vanish. Figure 4b displays additional orientation-amplitude results for systems with alternative feedback definitions that may be similarly understood.

4. Conclusions

To summarize, we numerically investigated the nonlinear supratransmission phenomenon in active 1D/2D periodic networks characterized by a non-local feedback control. We found that such a feedback adjusts the imaginary wavenumber component across the

whole of the frequency range, although asymmetrically, such that the otherwise reciprocal dynamics becomes direction-dependent. In the context of band gap energy transmission in finite, one-dimensional networks, the critical amplitude for stimulating supratransmission via boundary excitation differs for each boundary. Similarly, for a two-dimensional system, the amplitude threshold changes with boundary orientation. These results demonstrate an alternative, extremely tunable approach toward non-reciprocal dynamics and applications [9,14].

There are several research directions for subsequent studies. The influence of anisotropy and lattice type, especially those with multiple elements per lattice site (e.g., Lieb, Kagome, etc.), on the directionality of supratransmission is worth investigating. While we have considered a proportional controller with displacement inputs, a natural extension of the present work would explore controllers of derivative- and integral-type to determine the characteristics of each feedback archetype, independently followed by in combination. In addition, feedback definitions involving spatial and temporal rates have yet to be explored.

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