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Characteristic Analysis of DFIG Wind Turbine under Blade Mass Imbalance Fault in View of Wind Speed Spatiotemporal Distribution

Shuting Wan, Kanru Cheng , Xiaoling Sheng * and Xuan Wang

Department of Mechanical Engineering, North China Electric Power University, Baoding 071003, China

* Correspondence: sssxlyc@163.com

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Abstract: The blade mass imbalance fault is one of the common faults of the DFIG (Doubly-Fed Induction Generator) wind turbines (WTs). In this paper, considering the spatiotemporal distribution of natural wind speed and the influence of wind shear and tower shadow effect, the influence of blade mass imbalance faults on the electrical characteristics of DFIG WT is analyzed. Firstly, the analytical expressions and variation characteristics of electromagnetic torque and electromagnetic power under blade mass imbalance are derived before and after consideration of the spatiotemporal distribution of wind speed. Then simulations on the MATLAB/Simulink platform were done to verify the theoretical analysis results. The theoretical analysis and simulation results show that, considering the spatiotemporal distribution of wind speed and the influence of wind shear and tower shadow effect, the blade mass imbalance fault will cause fluctuation at the frequency of 1P (P = the frequency of rotor rotation), 3P, and 6P on electromagnetic power. Fluctuation at 1P is caused by mass imbalance while fluctuation at 3P and 6P are caused by wind speed spatiotemporal distribution; the amplitude of fluctuation at 1P is proportional to the degree of the imbalance fault. Since the equivalent wind speed has been used in this paper instead of the average wind speed, the data is more suitable for the actual operation of the WT in the natural world and can be applied for fault diagnosis in field WT operation.

Keywords: doubly-fed induction generator; blade mass imbalance; electrical characteristic; equivalent wind speed

1. Introduction

The blade mass imbalance of WT is a kind of fault that often occurs in the operation of WT. WT mostly operate in outdoor or offshore environments where the environment is relatively harsh, and sand or ice deposits may cause blade mass imbalance. In addition, fatigue stress will cause cracks in the blade, and the entry of dust, debris, rainwater, etc., into cracks, can cause blade mass imbalance, further aggravating the fatigue of the blade and other components. This will cause serious damage to the wind turbine generators (WTGs) if continued operation is allowed [1–3]. With the rapid global development of large-capacity WT, the occurrence of blade mass imbalances will become frequent. Therefore, the early identification and diagnosis of blade mass imbalance is an important problem to be solved in the field of fault diagnosis for WT [4,5]. Aiming for the identification and diagnosis of blade mass imbalance, the WT model was established in [6], and the finite element method was used to analyze the vibration characteristics of the WTGs caused by blade mass imbalance. Previous papers [7,8] collected the vibration signal of the WTGs on the simulation test-bed for spectrum analysis. The results show that the vibration of the WTGs in the horizontal direction will increase under the condition of mass imbalance. Previous studies have [9–11] analyzed the relationship between blade mass imbalance and generated power and have found that the blade mass imbalance fault will cause

fluctuation in output torque of the wind rotor, which is then reflected in the electric power signal of WTGs through the transmission chain. One study [9] achieved the preliminary diagnosis of blade mass imbalance and blade aerodynamic asymmetry by analyzing the power spectral density of WT. Another study [10] believes that the periodic variation of unbalanced forces caused by the blade mass imbalance will directly cause fluctuations of output power, but the reasons are not analyzed in detail. In a separate study [12], the relationship between generator stator current and shaft rotational frequency was analyzed under the condition of blade mass imbalance in direct-drive permanent-magnet WTGs. The stator current signal is used to analyze the change in characteristics of frequency to diagnose blade mass imbalance. In [13], the effects of blade mass imbalance and aerodynamic asymmetry on the stator current of the direct-drive permanent-magnet WTGs were analyzed, and the stator current square signal obtained by fast Fourier transform (FFT) was used for spectral analysis. The blade mass imbalance and winding asymmetry are diagnosed using the frequency, variation, and amplitude characteristics of the stator current square signal. In [14], the characteristics and changes in the time-domain, spectrum curve of rotor speed, and electromagnetic power signal—under different degrees of unbalanced faults—were analyzed. The results showed that electromagnetic power signal contained the first-harmonic component and second-harmonic component of the motor rotation, and amplitude increased as the unbalanced mass increased. [15] was simulated in MATLAB/Simulink, and the output power of the DFIG WTGs fluctuated when the blade mass imbalance was verified by the experiment. The frequency is the rotating frequency and its frequency of blade and doubling.

Through the analysis of existing research, most of the current research on WTGs are based on the average wind speed at the hub center, without considering the spatiotemporal distribution of wind speed. However, due to the presence of wind shear and tower shadow effects, the wind captured by WTGs in nature varies with time and space, and complex wind conditions can affect the output power of the WT. Scholars have discovered the phenomenon of periodic power fluctuation caused by wind shear and tower shadow effect for a long time. In [16], an analytical formulation of the generated aerodynamic torque of a three-bladed wind turbine, including the effects of wind shear and tower shadow, was presented. The model proves the existence of a 3P pulsation due to wind shear. In [17], a simulation system—supported by TurSim, AeroDyn, and FAST—was established, and the impact of wind shear, tower shadow effect, and yaw error on power generation quality were studied. In [18], the existence of the tower shadow effect and its influence on the output power was verified by analyzing the measured data of the wind farm. In [19], the effects of wind shear and tower shadow effects were studied via both power fluctuation and average output power. They concluded that both wind shear and tower response are sources of periodic power fluctuations and average power loss.

However, from current research, there has not been enough study of fault characteristics considering the spatiotemporal distribution of wind speed. Therefore, in the actual application of fault diagnosis, the diagnosis result may not match the ideal diagnosis result or may have a large diagnostic error, resulting in the failure of the diagnostic method. On the basis of considering the spatiotemporal distribution of practical natural wind speeds, this paper studies the electrical characteristics and the fault diagnosis of the DFIG WTGs when there is blade mass imbalance, and finds the fault diagnosis basis to assist the early detection and diagnosis of the fault. This may help in the avoidance of major accidents and improves the operating efficiency of DFIG WTGs.

2. DFIG WTGs Model and Blade Mass Imbalance Model

2.1. The Rotor Torque when Blade Mass Imbalance

The research object of this paper is the 3-blade DFIG WT; Figure 1 is a schematic diagram of the 3-bladed WT. Each blade can be equated to a mass block with a distance of r_i ($i = 1, 2, 3$) to the hub center and a mass of m_i ($i = 1, 2, 3$). In Figure 1, G_i and F_i ($i = 1, 2, 3$) represent the gravity and centrifugal force of each blade, respectively. Under normal conditions, these three blades are equal in gravity and the centrifugal force. Since the centrifugal force intersects the axis of the hub center

perpendicularly, the rotational torque of the rotor is not affected. The three blades have geometrical symmetry, so when the rotor rotates at angular velocity, ω_w , the torque generated by the gravity of the three blades is zero:

$$m_1 g r_1 \sin(\omega t) + m_2 g r_2 \sin\left(\frac{2}{3}\pi + \omega t\right) + m_3 g r_3 \sin\left(\frac{4}{3}\pi + \omega t\right) = 0 \quad (1)$$

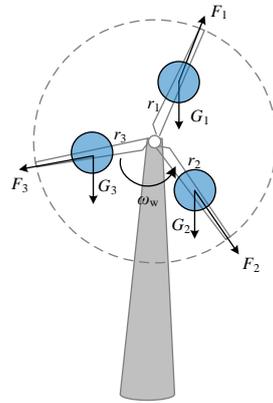


Figure 1. 3-bladed wind turbine schematic diagram.

Under normal conditions, the torque of the drive transmission chain is completely derived from the aerodynamic torque. In this paper, the aerodynamic torque model is established by the WT characteristic curve in [11], as shown in Equation (2):

$$\begin{cases} T_{w_0} = 0.5\pi\rho R^3 V_H^2 \frac{C_p(\sigma, \lambda)}{\lambda} \\ C_p(\sigma, \lambda) = 0.22\left(\frac{116}{\lambda_i} - 0.4\sigma - 5\right) \exp\left(-\frac{22.5}{\lambda}\right) \\ \frac{1}{\lambda_i} = \frac{1}{\lambda + 0.008\sigma} - \frac{0.035}{\sigma^3 + 1} \quad \lambda = \frac{\omega_w R}{v} \end{cases} \quad (2)$$

where T_{w_0} is the torque of the rotor (N·m); ρ is the air density (kg/m^3); R is the rotor radius (m); V_H is the wind speed at the height of the hub center (m/s); σ is the blade pitch angle ($^\circ$); λ is the tip-speed ratio; $C_p(\sigma, \lambda)$ is the rotor power coefficient; ω_w is the rotor angular velocity (rad/s). The presence of a trailing edge flap and the flapping angle may result in an aerodynamic torque imbalance, but its effect on torque is much less than the effect of mass imbalance on torque. In order to simplify the calculation process, their effect on torque is not considered when building the model.

When the blade mass imbalance fault occurs, the mass of one or several blades increases, which can be equivalent to the presence of an unbalanced virtual mass block m [20] on the blade. The distance to the hub center is R ; the block rotates together with the blade at an angular velocity, ω_w ; as shown in Figure 2, the force that the mass block is subjected to during the rotation mainly includes its gravity, mg , and centrifugal force, F_m . Under the action of gravity and centrifugal force, the WT transmission system will generate vibration along the main shaft. Because the tower has relatively high rigidity in the vertical direction, it will mainly cause periodic vibration of the blade and other structures in the horizontal direction. The vibration frequency is the base frequency of the rotor. The gravitational torque generated by the equivalent mass block affects the output rotor torque, T_w [21], and can be expressed as:

$$T_w = T_{w_0} + mgR \sin(\omega_w t + \phi) \quad (3)$$

where ϕ is the initial position angle of the blade (rad).

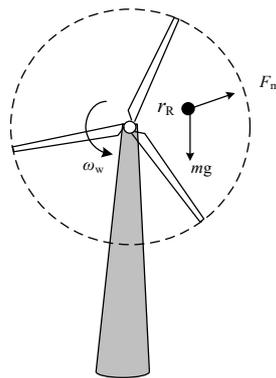


Figure 2. Force analysis of the imbalanced mass block.

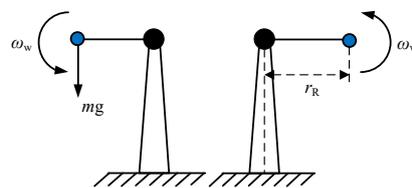


Figure 3. Model of blade mass imbalance.

Figure 2 is a schematic diagram of the mass imbalance of a single blade of a WT. The equivalent mass block, m , rotates with the blade at a speed of ω_w , and the gravitational torque generated by it can cause fluctuations in the rotational speed of the main shaft. As shown in Figure 3, the mass imbalance will accelerate the rotational speed of the main shaft during the downward rotation from the top to the bottom and will decelerate the main shaft as the blade rotates from the bottom up to the top.

2.2. The Electromagnetic Power when Blade Mass Imbalance

In order to obtain the electromagnetic power signal under the blade mass imbalance fault state, it is necessary to establish the model of transmission chain and generator. Considering that the flexibility of the transmission chain of the WT will affect the electromagnetic power, this paper will equvalate the transmission chain of WT into two mass blocks: the rotor mass block and the gearbox and generator mass block. The elasticity and damping of the shaft between different mass blocks are considered.

The transmission system of a WT is generally composed of a blade rotor, hub, gearbox, and generator. Ignoring the damping and elasticity between the blade and hub of the WT and regarding them as one mass block; and regarding the generator and the gearbox as another mass block, equivalates the WT transmission system to a two-mass system while the elasticity and damping of the shaft between the two masses are considered.

The DFIG WT connects the rotor and the generator through the speed increasing gearbox, and the angular velocity of the generator rotor is equal to $N\omega_w$ (N is the gearbox increase ratio). The increase and deceleration of the speed of the main shaft will be reflected according to the angular velocity of the generator rotor so that the generator rotor will have periodic fluctuations of speed increase and deceleration. The frequency is equal to the base frequency of rotor rotation. The two mass models can be expressed as:

$$\begin{cases} T_w - K_s\theta_s + B_s(\omega_e - \omega_w) = J_w \frac{d\omega_w}{dt} \\ -T_e + K_s\theta_s + B_s(\omega_w - \omega_e) = J_e \frac{d\omega_e}{dt} \end{cases} \quad (4)$$

where θ_s is the angle of torsion of the drive shaft (rad); ω_e is the rotational speed of the generator (rad/s); J_w and J_e are the rotational inertia of the rotor and the generator, respectively ($\text{kg}\cdot\text{m}^2$); K_s is the equivalent stiffness coefficient of the transmission chain; B_s is the damping coefficient; T_w is the aerodynamic torque of the rotor; T_e is the electromagnetic torque of the generator.

$$T_e = n_p L_m (i_{sq} i_{rd} - i_{sd} i_{rq}) \quad (5)$$

where n_p is the pairs of poles; L_m is the mutual inductance between stator and rotor coaxial equivalent windings in d - q coordinate system; i_{sd} , i_{sq} , i_{rd} , and i_{rq} are the d and q axis components of the stator and rotor current.

According to the knowledge of electrical engineering and combined with Equation (5), the electromagnetic power of the generator obtained is:

$$P_e = -\frac{3}{2} \omega_r L_m (i_{sq} i_{rd} - i_{sd} i_{rq}) = -\frac{3\pi f_1 T_e}{n_p} \quad (6)$$

where f_1 is the stator output rotational frequency.

So far, the DFIG model—including the rotor, the transmission chain, and the generator—has been established, and the torque generated by the blade mass imbalance has been introduced in the rotor part, so the response of the DFIG WT to the electromagnetic power under the fault can be simulated. Substituting these two equations into Equation (4) can establish the following relationship:

$$\left(\frac{J_w}{N} + J_e\right) \frac{d\omega_e}{dt} = T_w - T_e \quad (7)$$

The simultaneous Equations (2), (3), (5), and (6) are available:

$$P_e = -3\pi f_1 \frac{\left[T_{w0} + mgR \sin(\omega_w t + \phi) - \left(\frac{J_w}{N} + J_e\right) \frac{d\omega_e}{dt}\right]}{n_p} \quad (8)$$

It can be known from Equation (8) that the introduction of blade mass imbalance causes the electromagnetic power to show the frequency of rotational frequency, and its amplitude should be $A = -3\pi f_1 mgR/n_p$.

3. Influence of Wind Shear and Tower Shadow Effect on Electromagnetic Power

In the WT fault diagnosis analysis, the wind speed model is calculated by the wind speed, V_H , at the height of the hub center, and the change of the aerodynamic load on the blade does not be considered. In this paper, the equivalent wind speed model is used to simulate the influence of wind shear and tower shadow on mechanical torque by the wind speed fluctuation component; the equivalent wind speed model is proposed by Dale S.L.D. This model considers the wind shear effect and the tower shadow effect. The dynamic characteristics of the wind wheel are affected by the 3P pulsating component (3 is the number of blades, P is the rotor angular frequency) [16]. The wind speed in Equation (2) is replaced by the wind speed, V_H , at the hub to the equivalent wind speed V_{eq} , the equivalent wind speed, V_{eq} , expression is [22,23]:

$$V_{eq} = V_H + V_{ws} + V_{ts} \quad (9)$$

V_{ws} is the wind shear component (m/s); V_{ts} is the tower shadow effect component (m/s). They can be expressed as:

$$\begin{aligned}
 V_{ws} &= V_H \left(\frac{r \cos \beta + H}{H} \right)^\alpha \\
 &= V_H \left[\frac{\alpha(\alpha-1)}{8} \left(\frac{R}{H} \right)^2 + \frac{\alpha(\alpha-1)(\alpha-2)}{60} \left(\frac{R}{H} \right)^3 \cos 3\beta + \frac{\alpha(\alpha-1)(\alpha-2)(\alpha-3)}{576} \left(\frac{R}{H} \right)^4 \cos 4\beta \right]
 \end{aligned}
 \tag{10}$$

$$V_{ts} = \frac{MV_H}{3R^2} \sum_{b=1}^3 \left[\frac{A^2}{\sin^2 \beta_b} \ln \frac{R^2 \sin^2 \beta_b + x^2}{x^2} - \frac{2A^2 R^2}{R^2 \sin^2 \beta_b + x^2} \right]
 \tag{11}$$

$$\beta_1 = \beta \quad \beta_2 = \beta_1 + \frac{2\pi}{3} \quad \beta_3 = \beta_1 + \frac{4\pi}{3}
 \tag{12}$$

$$M = 1 + \frac{\alpha(\alpha-1)R^2}{8H^2}
 \tag{13}$$

where R is the rotor radius (m); H is the height of the hub center (m); β is the blade azimuth angle (rad), β_i ($i = 1,2,3$) is the blade azimuth angle of each blade ($^\circ$); α is the wind shear exponent; A is the tower radius (m); and x is the distance from the blade origin to the tower midline (m).

Equation (10) is obtained by expanding the Taylor series; the deviation resulting from the expansion items increases rapidly when the length of blade is larger than 60 m. Generally, it will expand to the third item in the actual computing application, and the fourth item will be ignored. When the length of blade reaches 100 m, under the condition that the azimuth angle of the blade is π , the third term deviation is about -11.5% , and the fourth term deviation is about -4.6% . As shown in Figure 4, it can be seen that with the increase of R , the fourth term in the expansion of the Taylor series can no longer be ignored as high-order infinitesimal, so this paper expands to the fourth term when building the model.

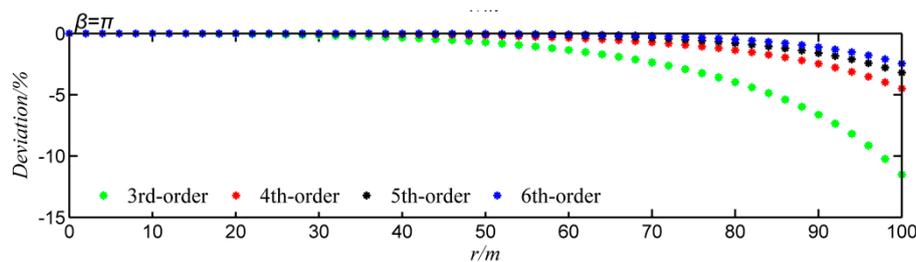


Figure 4. Deviations of high order components of Taylor series expansion.

Substituting Equations (10) and (11) into Equation (9), the equivalent wind speed obtained after simplification is:

$$\begin{aligned}
 V_{eq} &= V_H + V_H \left[M + \frac{\alpha(\alpha-1)(\alpha-2)}{60} \left(\frac{R}{H} \right)^3 \cos 3(\omega_w t) + \frac{\alpha(\alpha-1)(\alpha-2)(\alpha-3)}{576} \left(\frac{R}{H} \right)^4 \cos 4(\omega_w t) \right] \\
 &+ V_H \left\{ \frac{M}{3R^2} \sum_{b=1}^3 \left[\frac{A^2}{\sin^2 \beta_b} \ln \frac{R^2 \sin^2 \beta_b + x^2}{x^2} - \frac{2A^2 R^2}{R^2 \sin^2 \beta_b + x^2} \right] \right\}
 \end{aligned}
 \tag{14}$$

According to Equation (14), the equivalent wind speed of three different types of WTs at the hub wind speed $V_H = 11$ are plotted, and related parameters are shown in Table 1. As shown in Figure 5, these three curves clearly have a fixed fluctuate frequency.

Table 1. Parameter values of 3 different typical wind turbines.

P (kW)	R (m)	H (m)	A (m)	x (m)
1500	35	70	4.0/2.8 (3.4)	4.5
2500	56.5	87	4.6/3.2 (3.9)	5.2
8000	88.5	131	6.0/4.0 (5.0)	7.5

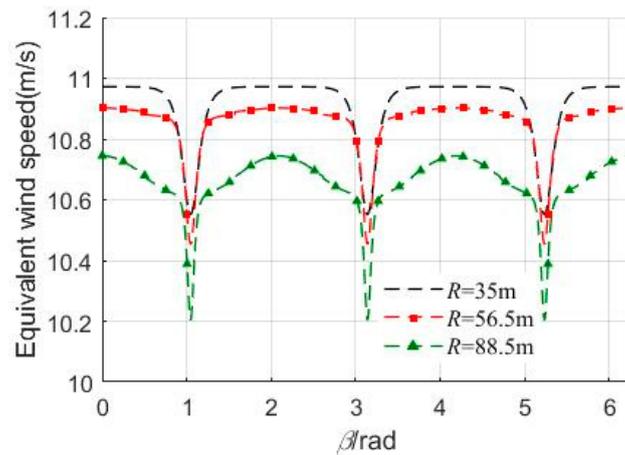


Figure 5. Equivalent wind speed corresponding to different wind turbines.

Analyzing Equations (10), (11), and (14), the amplitudes of the wind shear and tower shadow component in the equivalent wind speed are related to the WT structural parameters and are proportional to the hub wind speed, V_H ; the base frequency is 3 times the wind rotor rotational speed, also known as the 3P (P is the wind rotor rotation frequency) frequencies f_{3p} (Hz); the expression is:

$$f_{3P} = 3 \times \frac{n_\omega}{60} \quad (15)$$

where n_ω is the rotational speed of WT (r/min).

Replacing the wind speed at the hub center in Equation (2) with the equivalent wind speed including the wind speed at the hub, the wind shear component, and the tower shadow component, the aerodynamic torque of the WT under the equivalent wind speed is:

$$T_{w_0} = 0.5\pi\rho R^3 (V_H + V_{ws} + V_{ts})^2 \frac{C_p(\sigma, \lambda)}{\lambda} \quad (16)$$

Combined with the influence of blade mass imbalance in Section 2, the influence of electromagnetic power with the blade mass imbalance under equivalent wind speed is obtained. It can be expressed as:

$$P_e = -3\pi f_1 \frac{0.5\pi\rho R^3 (V_H + V_{ws} + V_{ts})^2 \frac{C_p(\sigma, \lambda)}{\lambda}}{n_p} - 3\pi f_1 \frac{mgR \sin(\omega_w t + \phi) - (\frac{J_w}{N} + J_e) \frac{d\omega_e}{dt}}{n_p} \quad (17)$$

It can be seen from the foregoing analysis that, due to the existence of wind shear and tower shadow effect, the wind speed contains a sinusoidal component whose frequency is three times the frequency of the frequency, and the component can be replaced by $k \sin(2\pi f_{3p} t + \phi)$; the part other than this component can be represented by Δv . Therefore, the equivalent wind speed can be rewritten as:

$$V_{eq} = V_H + k \sin(2\pi f_{3p} t + \phi) + \Delta v \quad (18)$$

Substitute Equation (18) into Equation (17) can get:

$$\begin{aligned} P_e &= -3\pi f_1 \frac{\left[0.5\pi\rho R^3 (V_H + k \sin(2\pi f_{3p} t + \phi) + \Delta v)^2 \frac{C_p(\sigma, \lambda)}{\lambda} + mgR \sin(\omega_w t + \phi) - (\frac{J_w}{N} + J_e) \frac{d\omega_e}{dt} \right]}{n_p} \\ &= -3\pi f_1 \frac{0.5\pi\rho R^3 [(V_H + \Delta v)^2 + 0.5k^2 (1 - \cos(2\pi f_{3p} t + \phi))] + k \sin(2\pi f_{3p} t + \phi) (V_H + \Delta v) \frac{C_p(\sigma, \lambda)}{\lambda}}{n_p} \\ &\quad - 3\pi f_1 \frac{mgR \sin(\omega_w t + \phi) - (\frac{J_w}{N} + J_e) \frac{d\omega_e}{dt}}{n_p} \end{aligned} \quad (19)$$

Equation (17) shows that the fluctuation of electromagnetic torque and electromagnetic power caused by blade mass imbalance fault and equivalent wind speed—including wind shear and tower shadow effect—will appear at 1P, 3P, and 6P; The fluctuation at 1P is caused by the blade mass imbalance, and the fluctuations at 3P and 6P are caused by the equivalent wind speed.

4. Simulation Results and Analysis

4.1. Simulation Model and Parameters

In software FAST of NREL (The National Renewable Energy Laboratory), there are wind speed models which include wind shear and tower shadow effects. The interface of this software is simple, clear, and easy to operate, but it is not convenient for fault simulation [24]. In order to verify the correctness of the theoretical analysis described above, a 1.5 MW DFIG WT under the blade mass imbalance fault condition was built in the MATLAB/Simulink environment. This adds in the equivalent wind speed model, including wind shear and tower shadow effects; the model is shown in Figure 6. The parameters of the WTs simulated in this paper are shown in Table 2, and the simulation is carried out at a rated wind speed of 11 m/s.

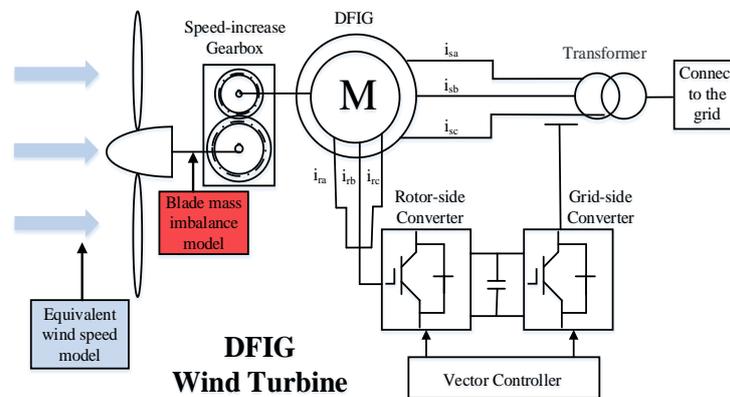


Figure 6. Simulation platform.

The model mainly includes the equivalent wind speed model, the output torque model under blade mass imbalance, the transmission chain model, and the generator model. By changing the weight of the unbalanced mass block, it is possible to simulate the influence of different degrees of mass imbalance faults on the electromagnetic torque and electromagnetic power. The rotor side of the DFIG WT adopts the stator voltage directional vector control [25,26].

Table 2. Parameters of 1.5MW DFIG wind turbine.

Parameters	Value	Parameters	Value
Rated power (MW)	1.5	Stator resistance (p.u.)	0.023
Rated speed (m/s)	11	Stator leakage inductance (p.u.)	0.18
Optimum tip speed ratio	8.7	Rotor resistance (p.u.)	0.016
Optimum rotor power coefficient	0.38	Rotor leakage inductance (p.u.)	0.16
Blade diameter (m)	35	Pairs of poles	3
Tower diameter (m)	1.7	Inertia constant (s)	0.685
Center height of the hub (m)	70	Air density (kg/m ³)	1.225

4.2. Fault Diagnosis Step

The blade mass imbalance fault diagnosis process of DFIG is shown in Figure 7 [27]. The choice of signal analysis method and diagnostic process are as follows:

Step1. The electromagnetic torque and electromagnetic power under blade mass imbalance condition were measured, and then spectrum analysis on the signal by FFT was performed to obtain the characteristic frequency and its amplitude as the reference value;

Step2. Electromagnetic torque and electromagnetic power were measured under the equivalent wind speed condition, blade mass imbalance condition, and then spectrum analysis was performed on the signal by FFT to obtain the characteristic frequency and its amplitude;

Step3. Comparing the amplitude of the characteristic frequency when the signal only exists, the blade mass imbalance diagnosis, the amplitude of the characteristic frequency of the signal at the imbalance condition under the equivalent wind speed, and the fault characteristics and frequency were obtained;

Step4. By comparing with the theoretical derivation results, the correctness of the conclusion in Section 3 was verified, and the fault characteristics summarized.

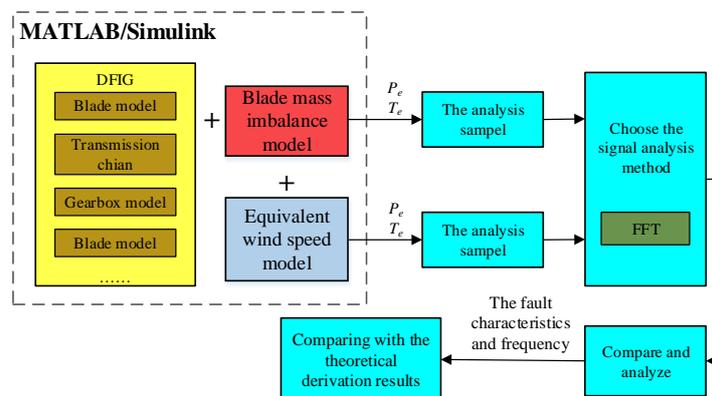


Figure 7. Flow diagram of fault diagnosis.

4.3. Fault Simulation Analysis

According to Sections 2 and 3, the blade mass imbalance and the equivalent wind speed can cause fluctuation in the output torque of the drive chain, which is reflected in the T_e and P_e at the generator. The correctness of the conclusions was verified through simulation and analysis. To quantitatively represent the degree of mass imbalance fault, the imbalance coefficient, b , was defined as:

$$b = \frac{T_{wg}}{T_{w0}} \times 100\% \tag{20}$$

To study the variation of the diagnosis characteristics under different degree of fault, b is set to 1%, 6%, and 10%, respectively. Figures 8 and 9 show the time-domain plot of T_e and P_e in normal conditions, only 10% mass imbalance, only the equivalent wind speed condition, and the equivalent wind speed, and 10% mass imbalance coexistence condition.

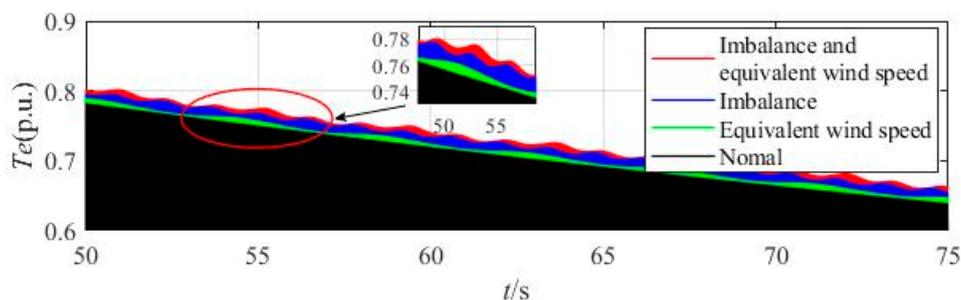


Figure 8. Electromagnetic torque time-domain waveform.

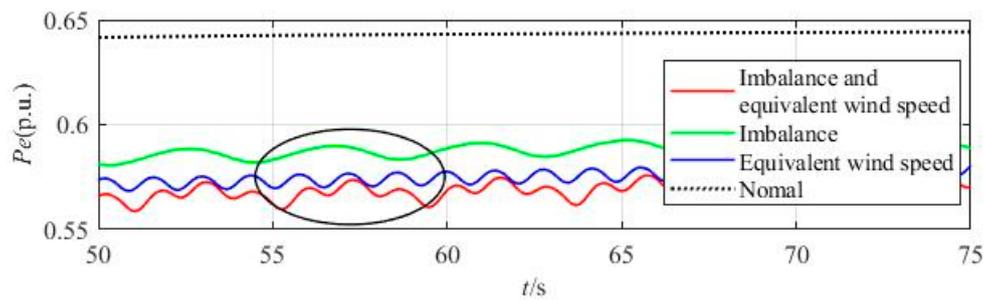


Figure 9. Electromagnetic power time-domain waveform.

It can be seen from Figures 8 and 9 that when the blade mass imbalance fault occurs, the T_e and P_e show obvious periodic fluctuation; when the blade mass imbalance occurs under the equivalent wind speed, there are two kinds of different fluctuation periods (as shown in the circled part of Figures 8 and 9). The longer period is roughly equivalent to the fluctuation period length of the equivalent wind speed; the shorter period is equivalent to the imbalance fluctuation period. Performing FFT processing on T_e and P_e , the spectrum diagram can be obtained, as shown in Figures 10 and 11.

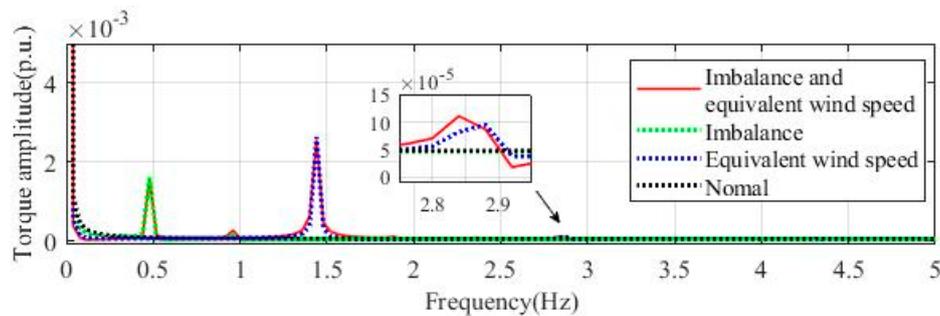


Figure 10. Spectrum of electromagnetic torque.

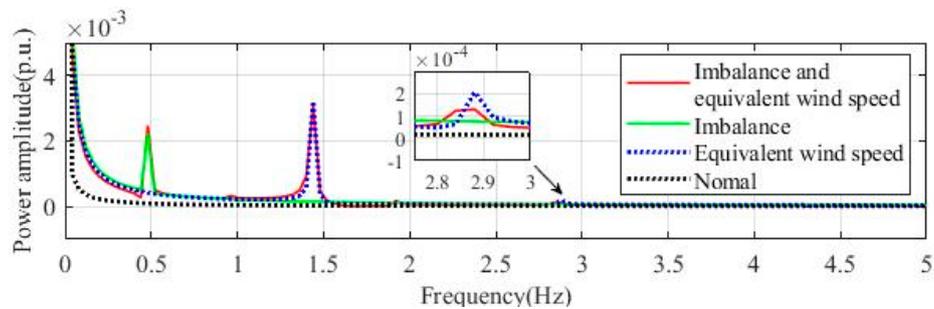


Figure 11. Spectrum of electromagnetic power.

It can be found from the T_e and P_e spectrum diagram that the T_e and P_e signal of the imbalance fault has a sharp peak at $f = 0.48$ Hz, which corresponds to the frequency of the rotor rotation. When magnified, the spectrum can find a smaller peak at $2P$; these conclusions are consistent with the conclusions in [11]. When the equivalent wind speed is added, it can be observed that, at $3P$ ($f = 1.44$ Hz), a sharp peak appears, and its amplitude is even slightly higher than $1P$; after the figure is partially enlarged, it can be observed that there is also a much smaller peak at $6P$ ($f = 2.88$ Hz), which results from the equivalent wind speed. Therefore, in the actual fault diagnosis, when the amplitude at $3P$ is greater than $1P$, it can also be judged that the blade mass imbalance fault has occurred.

Using the equation derived from the analysis, the calculation data of P_e at 6% imbalance condition under the equivalent wind speed was obtained, as was the spectrum curve. The spectrums obtained from the calculation data and the simulation data are drawn in the same figure, as shown in Figure 12. It can be seen that the two curves are very close, so it can be proven that the conclusions obtained by the simulation have a high reference value.

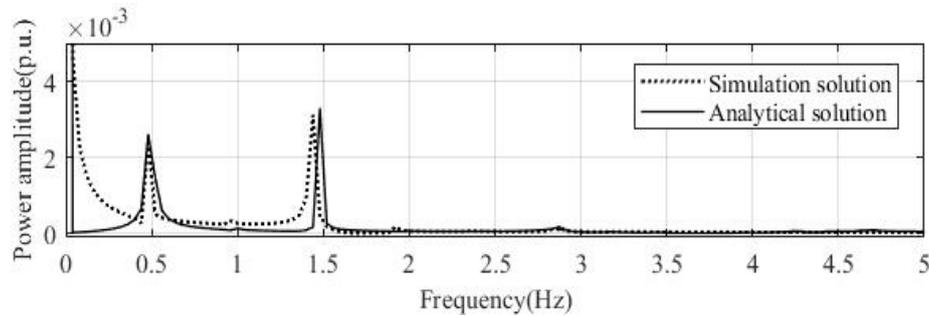


Figure 12. Spectrums obtained from the calculation data and the simulation data.

The electromagnetic power signals of the three different degrees of imbalance faults, at equivalent wind speed and the electromagnetic power signals and under normal conditions, are analyzed by FFT and plotted in the same figure. As shown in Figure 13, it can be seen that the greater the degree of fault is, the larger the amplitude at 1P, and the more obvious the fault characteristics. However, the amplitude at 3P and 6P does not change much with the increase of the degree of imbalance fault. Therefore, it is known that the faults at the 3P and 6P are mainly due to the existence of the equivalent wind speed, and this is the same as the conclusion drawn in the previous chapter.

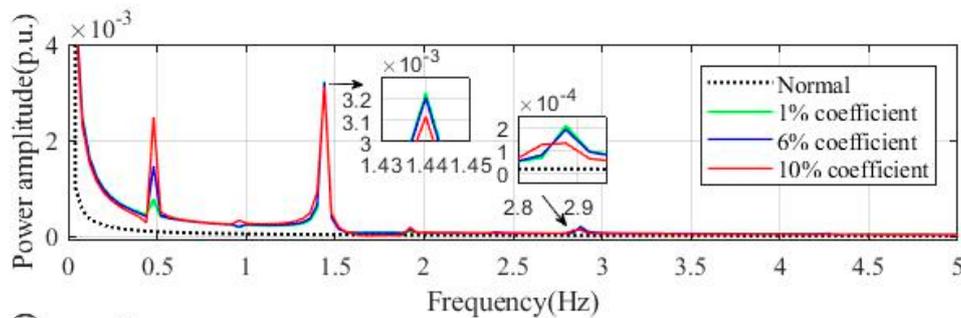


Figure 13. Electromagnetic power spectrum under $b = 1\%$, 6% , and 10% state.

The corresponding amplitudes at 1P, 3P, and 6P under different fault levels are listed, as shown in Table 3, and drawn as a line graph, as shown in Figure 14. It can be seen that as the degree of imbalance increases, the amplitude at 1P increases significantly, while the amplitude at 3P and 6P changes little, consistent with the foregoing conclusions.

Table 3. Amplitude of fault frequency for different degree of imbalance.

Imbalance Fault Degrees	Frequency (Hz)		
	0.48 Hz (f_{1P})	1.44 Hz (f_{3P})	2.88 Hz (f_{6P})
1%	0.000767	0.003222	0.000205
6%	0.001459	0.003203	0.000189
10%	0.002478	0.003116	0.000130

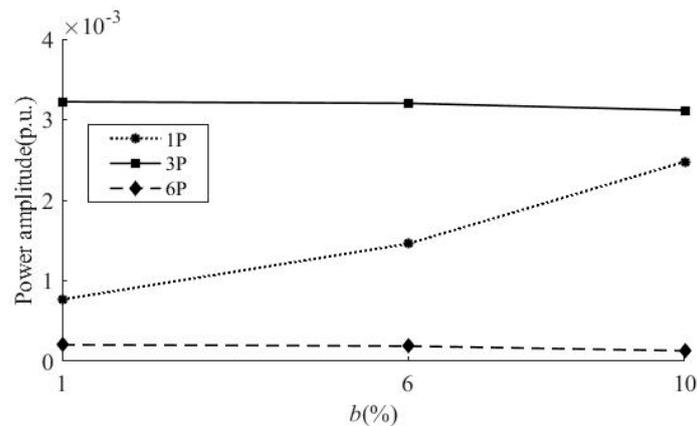


Figure 14. Amplitude of fault frequency for different degree of imbalance.

5. Conclusions

Based on the mechanism of blade mass imbalance, considering the spatiotemporal distribution of natural wind speed and the influence of wind shear and tower shadow effect, the equivalent wind speed model is established, and the electrical characteristics of the DFIG WT are analyzed. Via the theoretical and simulation analysis, the following conclusions were obtained:

- (1) The blade mass imbalance of the DFIG will cause fluctuations in the output torque of the wind rotor. The presence of wind shear and tower shadow effect will also cause fluctuations in the output torque and will pass through the transmission chain reflect in electromagnetic torque and electromagnetic power. So it is feasible to use an electrical signal to detect the fault.
- (2) Considering only the blade mass imbalance under the average wind speed at the hub will cause the electromagnetic torque and electromagnetic power to fluctuate at the 1P frequency.
- (3) Considering the spatiotemporal distribution of wind speed and the influence of wind shear and tower shadow effect, the blade mass imbalance fault will cause fluctuation at the frequency of 1P, 3P, and 6P on electromagnetic power, in which the fluctuation at 1P is caused by the mass imbalance fault while the fluctuation at 3P and 6P are caused by wind shear and tower shadow effect; the amplitude of the fluctuation at 1P is proportional to the degree of the imbalance fault.

Due to the introduction of the equivalent wind speed model based on the original blade mass imbalance fault diagnosis, the fault diagnosis method and process were improved, which provides a way for the diagnosis and identification of the blade mass imbalance fault in DFIG application. The signal is analyzed and processed by FFT in this paper. In future research, a variety of signal processing methods, such as wavelet transform, may be used to extract the fault characteristic frequency, and the extraction effect will be compared to find the most suitable signal processing method for blade mass imbalance fault diagnosis. In addition, this paper only discusses the failure situation of a single WT and does not consider other possible factors that may affect the wind speed in the wind farm, such as wake effects. In future research, these issues may be taken into consideration.

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Abbreviations

The following abbreviations are used in this manuscript:

b	Imbalance coefficient
B_s	Damping coefficient
$C_p(\sigma, \lambda)$	Rotor power coefficient
DFIG	Doubly-fed induction generator
f_1	Stator output rotational frequency
F_i	Centrifugal force of each blade
G_i	Gravity force of each blade
i_{rd}	d axis components of the rotor current
i_{rq}	q axis components of the rotor current
i_{sd}	d axis components of the stator current
i_{sq}	q axis components of the stator current
J_w	Rotational inertia of the turbine
J_e	Rotational inertia the generator respectively
K_s	Equivalent stiffness coefficient of the transmission chain
L_m	Mutual inductance
m_i	Mass of the equivalent mass block
n_p	Pairs of poles
n_ω	Rotational speed
P_e	Electromagnetic power
r_i	The distance between equivalent mass block to the hub center
R	Rotor radius
T_e	Electromagnetic torque of the generator
T_w	Aerodynamic torque of the rotor
T_{w_0}	Torque of the rotor
V_{eq}	Equivalent wind speed
V_H	Wind speed
V_{ts}	Tower shadow effect component
V_{ws}	Wind shear component
WT	Wind turbine
ω_e	Rotational speed of the generator
ω_w	Rotor angular velocity
λ	Tip-speed ratio
σ	Blade pitch angle
ρ	Air density
ϕ	Initial position angle of the blade
θ_s	Angle of torsion of the drive shaft

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