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# Complementary Power Control for Doubly Fed Induction Generator-Based Tidal Stream Turbine Generation Plants

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**Abstract:** The latest forecasts on the upcoming effects of climate change are leading to a change in the worldwide power production model, with governments promoting clean and renewable energies, as is the case of tidal energy. Nevertheless, it is still necessary to improve the efficiency and lower the costs of the involved processes in order to achieve a Levelized Cost of Energy (LCoE) that allows these devices to be commercially competitive. In this context, this paper presents a novel complementary control strategy aimed to maximize the output power of a Tidal Stream Turbine (TST) composed of a hydrodynamic turbine, a Doubly-Fed Induction Generator (DFIG) and a back-to-back power converter. In particular, a global control scheme that supervises the switching between the two operation modes is developed and implemented. When the tidal speed is low enough, the plant operates in variable speed mode, where the system is regulated so that the turbo-generator module works in maximum power extraction mode for each given tidal velocity. For this purpose, the proposed back-to-back converter makes use of the field-oriented control in both the rotor side and grid side converters, so that a maximum power point tracking-based rotational speed control is applied in the Rotor Side Converter (RSC) to obtain the maximum power output. Analogously, when the system operates in power limitation mode, a pitch angle control is used to limit the power captured in the case of high tidal speeds. Both control schemes are then coordinated within a novel complementary control strategy. The results show an excellent performance of the system, affording maximum power extraction regardless of the tidal stream input.

**Keywords:** Doubly-Fed Induction Generator (DFIG); energy harvesting; Maximum Power Point Tracking (MPPT); ocean energy; power control; power converters; Tidal Stream Turbine (TST)

## 1. Introduction

The growth of and need for renewable energy systems have led the European Commission to set a cost of energy reduction target of 0.20 euro/kWh by 2020 for wave and tidal energy [1]. However, analyzing current demonstration projects suggests that the Levelized Cost of Energy (LCoE) will not meet this target. The European Ocean Energy Forum has suggested, in its Draft Roadmap [2], that the cost of energy from wave and tidal farms could tend towards 0.10 euro/kWh by 2030 under the right conditions and through device deployment [3]. In this sense, renewable tidal energy systems are meant to produce electricity and ensure the cost reduction. To do so, these systems use power

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electronic converters in order to control and handle the flow of electrical power transferred to the grid. The use of the power converters is important since they represent the interface of the distributed power systems [4]. These power converters are optimally designed to assure maximum power production, provide reliable operation and ensure the protection of the power plant system [5]. Furthermore, the use of the Doubly-Fed Induction Generator (DFIG) associated with the power converters in a back-to-back configuration has been exploited for high power applications [6,7]. In this context, power electronics are changing the basic characteristics of the Tidal Stream Turbine (TST) from being an energy source to being an active power source for the grid. These devices enable efficient conversion of the variable frequency output of the generator, which is driven by a variable speed tidal turbine, to a fixed frequency appropriate for the grid [8]. The attractiveness of the tidal power stems from the huge energy potential and its predictable characteristics. The horizontal-axis turbines represent the most common type of tidal device, since horizontal-axis turbine devices account for 76% of research and development efforts in tidal devices over the world, as explained in [9]. The largest tidal power station in the world is Sihwa Lake situated in South Korea, which generates a peak rating of 254 MW, after La Rance Tidal Power Station in France with a total power output of 240 MW [10].

Tidal stream turbines are designed to extract the kinetic energy contained in the marine currents. These systems are intended to be installed in high energetic sites and are subjected to turbulence and strong waves where the force of the tidal current is large and the flow is highly variable [11,12]. Therefore, turbine design requires a precise knowledge of such high velocity flow in order to optimize the generated power output. This means that they need to be regulated, with a way of limiting the output power and shedding mechanical load at high flow speeds. In this context, the control of the generated power in a variable speed operation is ensured by the use of power electronic converters, while the blade pitch angle control is devoted to the limitation of the power in the case of the high tidal flow [13].

In the literature, the power control maximization is achieved by means of the DFIG through the back-to-back converter [14,15]. The Rotor Side Converter (RSC) control is used to keep the rotational speed of the generator at its optimal value and to minimize the core losses, while the Grid Side Converter (GSC) control aims to maintain the voltage of the DC-link and control the reactive power [16,17]. A variable speed control strategy of a Marine Current Turbine (MCT) with a fixed pitch angle was proposed in [18]. The results show that the proposed control strategy is effective in terms of speed tracking, and it has been analyzed with respect to the swell effect for the tidal input model. However, the active power exhibits some tracking errors. In power limitation mode, the control is ensured by the angular position of the blades according to the hydrodynamic power limitation targets. Two control strategies are investigated in [19]; the stall and pitch angle controls. A comparative study about stall and pitch control of a TST was discussed. The study suggests that the stall-regulated systems are not able to keep a constant power output in the case of strong tidal velocity. The pitch regulation control contributes to more efficient results regarding the energy yields. Several research works focused on the pitch angle control as discussed in [20,21]. To overcome the drawbacks of a fixed pitch angle control, a novel complementary control is presented in this paper. This control combines both aforementioned strategies to provide a suitable switching tool for efficient and robust energy tidal conversion against different tidal velocities. This is achieved by the proposed switching algorithm, ensuring a smooth transition between variable speed mode and power limitation mode.

In this paper, the modeling and control of a TST-based DFIG and AC-DC-AC Pulse Width Modulation (PWM) power converter are presented. The novel control strategy has a valuable role in order to improve all aspects of the tidal stream generator and a strong influence on the dynamic behavior of the system. This study provides evidence that operating the turbine in variable speed mode will increase the energy yield of the turbine by allowing it to work at its maximum power coefficient over a wide range of tidal speeds. In the case of strong tidal currents, the pitch regulation is conceived to afford the excess power and keep the turbine operating within the specified limits. The goals, when controlling a variable-speed TST , are optimizing the harnessed energy, regulating the generated

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power and reducing mechanical loads. Therefore, it is obvious that there is a need to improve the efficiency of the studied system and the produced power potential. Below rated flow speed, the turbine is adjusted to keep the tip speed ratio at its optimal value. To do so, a voltage oriented strategy is used to regulate the GSC, and a stator flux control is applied in the RSC including the rotational speed and the rotor currents control loops. A rotational speed control based on a proportional integral controller is used to track the optimal regime characteristic by implementing a Maximum Power Point Tracking (MPPT) approach. At high tidal speed, the mechanical control of the blade pitch angle is set using a proportional controller to control the pitch actuator in order to limit the active power. A switch controller is implemented based on a novel algorithm developed to ensure a smooth transition between both operation modes.

The remainder of this paper is organized as follows. Section 2 introduces the theory behind tidal power. The hydrodynamic turbine model, the DFIG and the back-to-back converter are presented in Section 3. Section 4 discusses the control statement detailing two control strategies that are proposed for the rotational speed and pitch angle controls. Then, it deals with the novel complementary control, which combines both control schemes. Three demonstrative study cases are presented and discussed in Section 5. Finally, concluding remarks are drawn in Section 6.

## 2. Background on Tidal Power

Tidal currents are generated from the ebb and flow of the tides. Besides, these currents are affected by the climate influences and the weather disturbances [22]. Under the effect of gravitational attraction, the Earth is both attracted and repulsed by the Moon. As illustrated in Figure 1, when the Sun and the Moon are orthogonal to the Earth, their impacts oppose each other: these are the neap tides. Whereas, when the axes are aligned, the Sun amplifies the impact of the Moon, generating the spring tides [23].

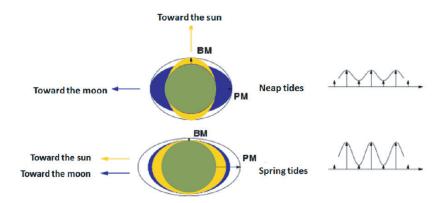


Figure 1. Effect of the Sun and the Moon on the tides.

Tidal currents are predictable in time and in amount of current velocity. This feature is extremely essential for a successful integration of the turbines with the grid [24]. For this kind of energy, the technology of the tidal turbine is very mature and has helped to advance its technological readiness level [25]. TST systems are installed in locations with high tidal current speed or strong continuous ocean currents. Lying down at the bottom of the seabed, they extract power from the running water. Tidal velocity may be measured using an Acoustic Doppler Current Profiler (ADCP) at a specific time period and water depth [26].

There are many areas of the world where extreme tidal currents may be observed, but the waveform of tidal velocity varies at different locations of the Earth. Among myriad tidal speed profiles, Figure 2 shows a semidiurnal tide with monthly variations, which is found in Raz de Sein Brittany, France. This type of tide, which consists of spring and neap tides, has a period of, approximately, 12 h and 25 min [27].

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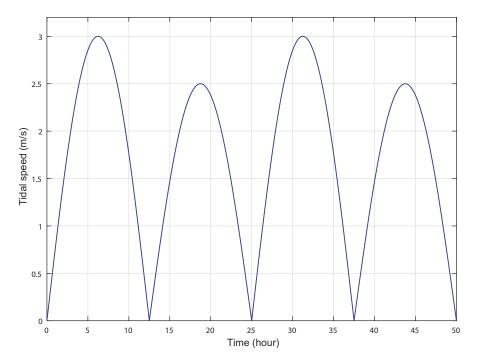


Figure 2. Tidal velocity variation per hour.

The flux of kinetic energy in a tidal current is related to the speed of water passing through the cross-section of this channel. It is expressed in  $(W/m^2)$  as given by [28] as:

$$P = \frac{1}{2}\rho \int_{\Lambda} \frac{V^3 dA}{A} \tag{1}$$

where  $\rho$  is the fluid density (kg/m<sup>3</sup>), A is the cross-sectional area of the turbine rotor (m<sup>2</sup>) and V is the component of the current flow velocity perpendicular to the cross-section of the channel (m/s) [28].

Equation (1) can be modified to allow the definition of a convenient average velocity  $V_A$ , across the cross-section of the channel. The simplification is expressed as:

$$AV_A^3 = \int_{\Delta} (V^3 dA) \tag{2}$$

The provided flux density variation with  $V_A$  is depicted in Figure 3, which shows the influence of the flow speed on kinetic energy [29].

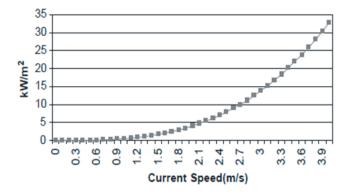


Figure 3. Flux density versus current speed.

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#### 3. System Description

The configuration of a DFIG-based TST system is shown in Figure 4. The DFIG is essentially a wound rotor induction generator in which the rotor circuit can be controlled by external devices to achieve variable speed operation. The stator of the generator is connected to the grid through a transformer, whereas the rotor connection to the grid is done through AC-DC-AC power converters.

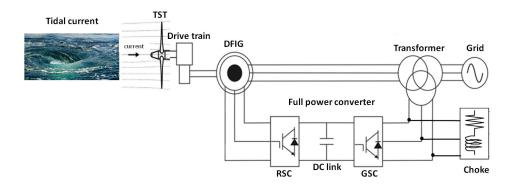


Figure 4. General scheme for a DFIG-based TST system.

## 3.1. Hydrodynamic Turbine Model

The amount of power captured from a tidal turbine is governed as follows [30]:

$$P_t = \frac{1}{2} C_p(\lambda, \beta) \rho \pi R^2 V^3 \tag{3}$$

where  $P_t$  is the turbine power (W), R is the radius of the turbine blades (m) and  $C_p$  is the power coefficient. The expression of  $C_p$  is obtained by an approximating function, which depends on the blades pitch angle  $\beta$  (°) and the tip-speed ratio,  $\lambda$ , defined as follows [31]:

$$\lambda = \frac{\omega_t R}{V} \tag{4}$$

where  $\omega_t$  is the rotational speed of the rotor (rad/s).

The hydrodynamic torque, expressed in (Nm) and developed by the tidal turbine, is given as follows:

$$T_{tst} = \frac{P_t}{\omega_t} \tag{5}$$

#### 3.2. Drive-Train Two-Mass Model

The hydrodynamic torque, generated by the rotation of the rotor, is transferred to the generator via the drive train [32]. In order to model the drive-train, a two-mass model is used whereby the rotor and the generator are connected together via a flexible shaft, which has stiffness  $K_{sh}$  (Nm/rad) and damping  $D_{sh}$  (Nms/rad) coefficients. The dynamics of the drive-train two-mass is expressed as follows:

$$T_{tst} - T_t = 2H_t \frac{d\omega_t}{dt} \tag{6}$$

$$T_t = D_{sh}(\omega_t - \omega_g) + K_{sh} \int (\omega_t - \omega_g) dt$$
 (7)

$$T_t - T_{em} = 2H_g \frac{d\omega_g}{dt} \tag{8}$$

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where  $T_t$  is the mechanical torque from the generator shaft (Nm),  $T_{em}$  is the generator electromagnetic torque (Nm) and  $\omega_g$  is the generator rotational speed (rad/s).  $H_t$  and  $H_g$  are the inertia constants expressed in (s) for the turbine and generator, respectively.

## 3.3. Doubly-Fed Induction Generator Model

The DFIG-based TST will offer many advantages, such as the ability to generate power at variable speed based on four quadrants' active and reactive power capabilities [33,34]. Furthermore, the DFIG is robust and requires little maintenance [35]. For the proposed control strategy, the generator's dynamic model is defined in the synchronous d-q frame using Park's transformation as detailed in [36]. The expressions of the stator and rotor d-q axis voltages in (V) are given as follows:

$$\begin{cases}
U_{sd} = R_s I_{sd} + \frac{d\varphi_{sd}}{dt} - \omega_s \varphi_{sq} \\
U_{sq} = R_s I_{sq} + \frac{d\varphi_{sq}}{dt} - \omega_s \varphi_{sd} \\
U_{rd} = R_r I_{rd} + \frac{d\varphi_{rd}}{dt} - \omega_r \varphi_{rq} \\
U_{rq} = R_r I_{rq} + \frac{d\varphi_{rd}}{dt} - \omega_r \varphi_{rd}
\end{cases} \tag{9}$$

The direct and quadrature components of the stator and rotor flux in (Wb) are defined as:

$$\begin{cases}
\varphi_{sd} = L_{s}I_{sd} + L_{m}I_{rd} \\
\varphi_{sq} = L_{s}I_{sq} + L_{m}I_{rq} \\
\varphi_{rd} = L_{r}I_{rd} + L_{m}I_{sd} \\
\varphi_{rq} = L_{r}I_{rq} + L_{m}I_{sq}
\end{cases} (10)$$

The DFIG electromagnetic torque is expressed in the *d-q* frame as:

$$T_{em} = \frac{3}{2}pL_m \left(I_{sq}I_{rd} - I_{sd}I_{rq}\right) \tag{11}$$

where  $I_{sdq}$ ,  $I_{rdq}$  are the stator and rotor d-q axis currents (A) ,  $\omega_s$ ,  $\omega_r$  are the stator and rotor pulsations (rad/s),  $R_s$  and  $R_r$  are the stator and rotor resistances ( $\Omega$ ),  $L_s$  and  $L_r$  are the stator and rotor inductances (H) , respectively,  $L_m$  represents the magnetizing inductance (H) and p is the pole pair number.

#### 3.4. Back-To-Back Converters

The use of two six switch-based converters connected in a back-to-back configuration with an intermediate DC-link capacitor has the advantage of allowing for vector control on both the generator and grid-side converters [36]. The used full AC-DC-AC power converter consists of an RSC and a GSC. The RSC is intended to control the operation of the generator. According to the dynamical model of the DFIG, which is defined in the d-q frame, the vector control strategy is used. The objective of the GSC is to keep the DC-link voltage constant regardless of the magnitude and direction of the rotor power. A vector control strategy is employed to control the reactive power [37].

The exchanged active and reactive power in (W) and (VAR), respectively, of the system can be calculated by:

$$P_g = \frac{3}{2} \left( U_{dg} I_{dg} - U_{qg} I_{qg} \right) \tag{12}$$

$$Q_g = \frac{3}{2} \left( U_{qg} I_{dg} - U_{dg} I_{qg} \right) \tag{13}$$

where  $U_{dg}$ ,  $U_{qg}$  (V) and  $I_{dg}$ ,  $I_{qg}$  (A) are the grid voltages and currents in the d and q reference frame, respectively.

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The *d*-axis of the synchronous frame and the grid voltage vector are aligned ( $U_{dg} = U_g$ ) and ( $U_{qg} = 0$ ) in order to achieve the voltage oriented control. Therefore, the expressions of the active and reactive power are as follows:

$$P_g = \frac{3}{2} U_g I_{dg} \tag{14}$$

$$Q_g = -\frac{3}{2} U_g I_{qg} \tag{15}$$

The relationship between the power stored in the DC-link and the power flowing to the grid can be given as:

$$P_g = \frac{3}{2} U_g I_{dg} = U_{dc} i_{dc} \tag{16}$$

where  $U_{dc}$  (V) and  $i_{dc}$  (A) are the voltage and current across the DC-link, respectively.

# 4. Control Objectives and Strategies

The proposed complementary control strategy describes how the TST plant is handled to maximize the obtained power output. This is achieved by adequately adjusting the rotational speed of the rotor at a steady-state for each tidal speed input within the range of the turbine operation. In the operation of a variable speed, depending on the value of the nominal tidal speed, two regions are distinguished. On the one hand, when the plant is operating below nominal power, the main control objectives are maximizing the power captured from the tides and minimizing the loads submitted by the drive-train shaft; whereas, when operating in the full load regime, it is necessary to limit the amount of captured power to avoid the generator overload. To do so, the rotational speed is kept constant at its nominal value, and the captured turbine power must be accordingly regulated. The use of the AC-DC-AC power converters is the key to achieve the control objectives. In this sense, the RSC is used to implement the rotational speed control scheme that allows maximizing the power extraction by regulating the turbine's rotational speed [38]. Meanwhile, the GSC is used to maintain the DC-link voltage constant and to compensate the reactive power. On the other hand, pitch angle control is used when the system is working in power limitation mode to govern the pitch actuator in order to limit the active power and ensure the survivability of the turbine. A proportional controller is set to find the adequate pitch angle for which the power is maintained at its nominal value. The output of the controller will serve as the control signal to the actuator allowing it to rotate the blades to the desired angular position. To alternate between both operation modes, a complementary control strategy is implemented. This is achieved by a proposed switching algorithm, ensuring a smooth transition between both control schemes. The proposed control scheme for the DFIG-based TST system is illustrated in Figure 5.

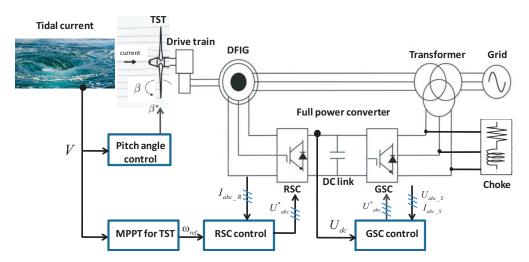


Figure 5. System control description.

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#### 4.1. GSC DC-Link Control Design

The voltage oriented strategy is used to control the grid-connected inverter. The main objective is to maintain the exchange power to the generator with the grid [39]. The proposed scheme of the implemented control approach is shown in Figure 6.

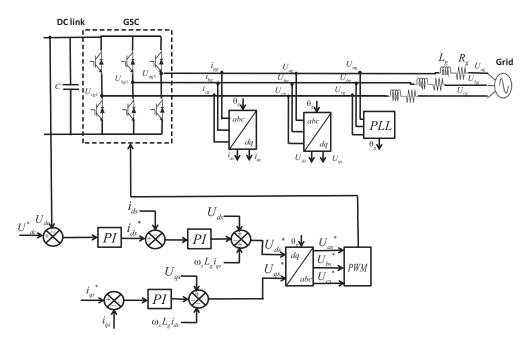


Figure 6. Voltage-oriented control structure for the Grid Side Converter (GSC).

In such a control scheme, a Phase Locked Loop (PLL) block is used to recover the phase of the input signal which, denotes  $\theta_g$ . The d-q axis currents and voltages are obtained using Park's transformation. The grid voltages (V) can be defined as follows:

$$\begin{cases}
U_{ag} = i_{ag}R_g + L_g \frac{di_{ag}}{dt} + U_{ag1} \\
U_{bg} = i_{bg}R_g + L_g \frac{di_{bg}}{dt} + U_{bg1} \\
U_{cg} = i_{cg}R_g + L_g \frac{di_{cg}}{dt} + U_{cg1}
\end{cases}$$
(17)

where  $R_g$  ( $\Omega$ ) and  $L_g$  (H) are the grid coupling resistance and inductance, respectively,  $U_{ag1}$ ,  $U_{bg1}$ ,  $U_{cg1}$  (V) are the three phase converter terminal voltages and  $i_{ag}$ ,  $i_{bg}$ ,  $i_{cg}$  (A) are the three phase grid currents. Using Park's transformation, Equation (17) can be written in the d-q reference frame as follows:

$$\begin{cases}
U_{gd} = i_{ds}R_g + L_g \frac{di_{ds}}{dt} - \omega_s L_g i_{qs} + U_{gd1} \\
U_{gq} = i_{qs}R_g + L_g \frac{di_{qs}}{dt} - \omega_s L_g i_{ds} + U_{gq1}
\end{cases}$$
(18)

According to Equations (14) and (15), the active and reactive power are controlled via the d-axis and q-axis current, respectively. The current control loops are identical and generate the grid voltage references  $U_{ds}^*$  and  $U_{qs}^*$  as:

$$\begin{cases} U_{gd}^* = U_{gd} + \Omega_g L_g i_q - (K_{Pi} e_d + K_{Ii} \int e_d dt) \\ U_{gq}^* = U_{gq} - \Omega_g L_g i_d - (K_{Pi} e_q + K_{Ii} \int e_q dt) \end{cases}$$
(19)

where  $\Omega_g$  is the synchronous frequency (rad/s),  $K_{Pi}$ ,  $K_{Ii}$  are the gains of the current Proportional Integral (PI) controller and  $e_d$ ,  $e_q$  are the errors of the currents defined in the d-q reference frame, respectively. In order to enhance the transient response of the system, compensator terms and

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feed-forward voltage are added to the control signals [40]. Finally, the reference voltages transformed to the three-phase abc frame are then used to generate all PWM signals for the GSC.

There are three feedback control loops; one outer voltage loop for the control of DC voltage  $U_{dc}$  and two inner currents loops for the control of the direct and quadrature axis currents  $i_{ds}$  and  $i_{qs}$ , respectively.  $i_{qs}$  is used to regulate the reactive power. During normal operation, the converter will transfer all of the active power generated by the TST to the grid. Thus, the q-axis current reference is set to zero.

The outer voltage control loop generates the current reference for the inner current loop. The inner loop must be faster to ensure that there is no interaction with the outer voltage loop. Thus, the inner loop can be approximated as a unity gain [40]. The closed loop scheme of the DC-link voltage is shown in Figure 7.

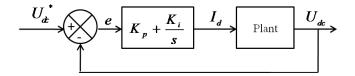


Figure 7. Outer DC-link voltage loop design.

Recalling Equation (16), the transfer function of the plant is expressed in the continuous domain as:

$$\frac{U_{dc}(s)}{I_d(s)} = \frac{3}{4} \frac{m_a}{Cs} \tag{20}$$

where  $m_a$  is the modulation ratio and C is the capacitance in (F).

The desired closed loop poles are characterized by their relative damping coefficient and their natural frequency. Assuring an overshoot of 5%, the damping coefficient is 0.69 as given in [41]. The natural frequency  $\omega_0$  (rad/s) becomes a closed-loop performance parameter that is specified according to the desired closed-loop response requirement. From the simulation of a step response, the settling time is estimated as:

$$t_s pprox rac{5\xi}{\omega_0}$$
 (21)

The inner current control loop design is presented by Figure 8. The converter model is described by a delay of two sample periods.

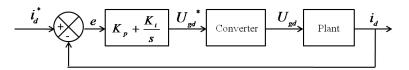


Figure 8. Inner current control loop design.

The transfer function of the plant can be modeled by first order system, which is given by Equation (22). A third order closed loop of the current loop is defined.

$$\frac{1}{L_{g}s + R_{g}} \tag{22}$$

The design of the DC-link voltage and currents controllers are carried out using the experimental method of Ziegler-Nichols [42]. After obtaining the initial values of the gains, further refinement were introduced based on the robust response time algorithm [43].

#### 4.2. RSC Rotational Speed Control Design

In order to capture the maximum power from tidal energy, we must permanently adjust the rotational speed of the turbine to the tidal velocity by means of the Rotor Side Converter (RSC). The maximum amount of energy extraction equals the 16/27th part of the kinetic energy in the current. This limit is often referred to as the Lanchester-Betz limit. Therefore, controlling the power coefficient by maintaining the tip speed ratio  $\lambda_{opt}$  at its optimal value is the main characteristic of the system [44]. For each tidal current speed, there is a certain rotational speed at which the power curve reaches its maximum value. All of these maxima compose what is known in the literature as the Optimal Regimes Characteristic (ORC) [45–47]. As shown in Figure 9, the maximum power for the turbine under study is  $P_n = 1.5$  MW at  $V_n = 3.2$  m/s.

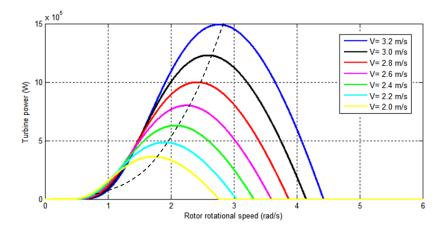


Figure 9. Power versus rotational speed for different tidal speeds.

Figure 10 shows how the turbine power varies with the rotational speed for different values of the pitch angle at the rated tidal speed  $V_n$ . The maximum power is reached for  $\beta = 0^\circ$ , and as  $\beta$  increases, the power decreases. This feature aims to avoid the TST mechanical overload when the tidal velocity is above the rated value.

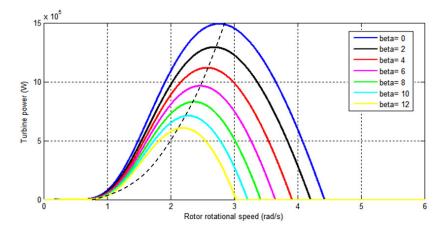


Figure 10. Power versus rotational speed for different pitch angles at rated tidal speed.

The proposed stator flux control strategy applied to the RSC is presented in Figure 11. The measured parameters are the stator and rotor currents, the stator voltage and the rotor speed. The controller structure consists of two inner current loops and an outer rotational speed loop.

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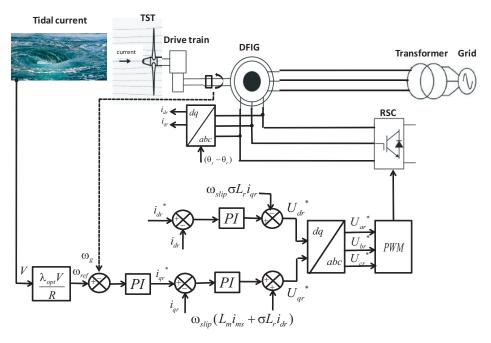


Figure 11. Stator flux-oriented control structure for the Rotor Side Converter (RSC).

The speed control loop design is conceived using the Maximum Power Point Tracking (MPPT) approach to control the generator rotational speed via the RSC and must be adjusted to track the optimal reference given as follows:

$$\omega_{ref} = \frac{\lambda_{opt} V}{R} \tag{23}$$

Equation (23) was used to determine the reference speed  $\omega_{ref}$  expressed in (rad/s) for the speed control loop, which defines a reference signal for the q-axis rotor current  $i_{qr}^*$ . The d-axis current reference  $i_{dr}^*$  is set to zero. The process will be controlled by a PI controller in the continuous domain; the transfer function of the plant model is retrieved from Equation (8). Similarly to the DC-link voltage controller for the GSC, the controller gains of the speed control loop are obtained using the experimental method of Ziegler-Nichols.

For the inner current control loops determine the d-q rotor voltage reference. The relationship between the rotor voltages (V) and the rotor currents (A) is as given in [34]:

$$\begin{cases}
U_{dr} = R_r i_{dr} + \sigma L_r \frac{di_{dr}}{dt} \\
U_{ar} = R_r i_{ar} + \sigma L_r \frac{di_{dr}}{dt}
\end{cases}$$
(24)

where  $\sigma$  is the leakage factor.

The current control loops are identical and generate the rotor voltage reference  $U_{dr}^*$  and  $U_{qr}^*$  as presented by Equation (25). Furthermore, decoupled terms are added to the rotor voltage references that will improve the transient response of the system [48].

$$\begin{cases}
U_{dr}^{*} = -\omega_{slip}\sigma L_{r}i_{qr} + (K_{Pi}e_{d} + K_{Ii}\int e_{d} dt) \\
U_{qr}^{*} = \omega_{slip}(L_{m}i_{m} + \sigma L_{r}i_{dr}) + (K_{Pi}e_{d} + K_{Ii}\int e_{d} dt)
\end{cases}$$
(25)

where  $\omega_{slip}$  is the slip angular frequency (rad/s) and  $i_m$  is the stator magnetizing current, which is considered constant.

The control scheme for the RSC currents loops are identical. The block diagram is similar to the one of the current loop for the GSC of Figure 8 and assuming that the current loop is faster than the outer speed loop. After analyzing the second order system, the step response analyses is used

to assess the closed-loop performance of the PI controller. The quadrature components of the rotor voltage reference are transformed to the abc stationary frame to be imposed to the RSC through the PWM block.

# 4.3. Pitch Angle Control

The control of the blade pitch is a useful method in order to avoid the generator overload when higher tidal speeds occur. A suitable pitch angle for stable operation can be generated by using the power error. To do so, the value of the pitch angle should increase when the turbine power is above the nominal value. In this mode of operation, the rotor rotational speed is maintained constant, and the turbine power is regulated by means of the pitch angle controller. The control scheme of the pitch angle controller is shown in Figure 12.

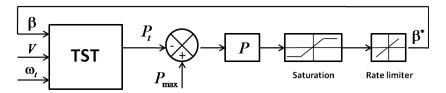


Figure 12. Block diagram of the pitch controller.

The pitch controller has the measured tidal turbine power  $P_t$  and the power reference  $P_{\text{max}}$  as inputs.  $P_{\text{max}}$  is set to the maximum generator power, which is equal to 1.5 MW. A proportional controller is set to find the pitch angle  $\beta$  from the difference between  $P_t$  and  $P_{\text{max}}$ . Then, a saturation block is added to limit the value of  $\beta$ . As shown in Figure 13, the pitch angle can vary over the range of  $\beta = [0^{\circ}, 21^{\circ}]$ . Moreover, the pitch angle's variation over time is limited. As for wind turbines, the pitch angle rate can vary between  $3^{\circ}$ /s and  $10^{\circ}$ /s, as shown in [49]. For this study, the rate pitch angle used is about  $10^{\circ}$ /s for large turbine size.

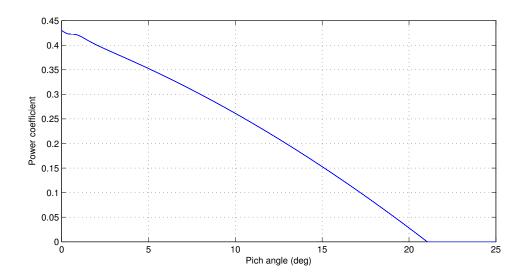


Figure 13. Power coefficient versus pitch angle.

#### 4.4. Complementary Control

The proposed complementary control consists of improving the performance and dynamic load assessment of the system under different operating conditions by adequately controlling the TST transition under and above the tidal current speed threshold value conditions. That is, the system is regulated so as to smoothly pass from the variable speed mode to the power limitation mode in

order to optimize the generated output power. To do so, a switching algorithm is used. The variable speed and the power limitation modes are operating when the tidal velocity is under and above high flow speed, respectively. In the case of variable speed mode, the rotational speed control by means of the RSC is activated while keeping the pitch angle constant in order to drive the system to maximum power. In turn, when the tidal speed is high, the pitch angle control is activated to adequately rotate the blades at the suitable angular position and thus maintaining the rotational speed constant to successfully assure the power limitation. When the speed of the tide is within the limits of these regions, the system can pass from one mode to another often enough, which conducts supplementary stress to the pitch angle actuator.

The proposed novel switching control block is depicted in Figure 14, where  $\omega_n$  is the rated generator rotational speed,  $\omega_{MPPT}$  is the reference speed from the MPPT block and  $\omega_{ref}$  is the reference rotational speed imposed on the RSC controller. The regulated pitch angle from the pitch controller block is denoted as  $\beta_{pitch}$ ,  $\beta_{min}$  is setting null and  $\beta_{ref}$  is the value of the pitch angle reference, which is attributed to the pitch actuator.

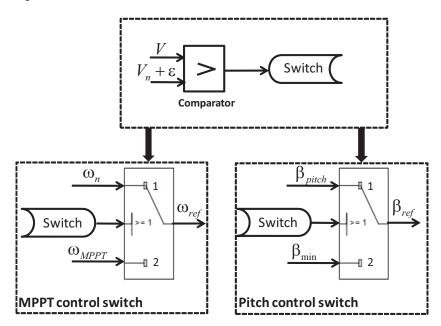


Figure 14. Block diagram of the switch controller.

In order to avoid frequent switching, we set a limit to the system as an interval  $[(V_n - \varepsilon), (V_n + \varepsilon)]$  instead of a fixed value  $V_n$ .  $\varepsilon$  represents the margin error of the measured tidal speed in order to ensure the protection of the turbine from overload. The comparator block compares V to  $V_n$  over an average time period  $T_{avr}$  and if the condition  $V > (V_n + \varepsilon)$  has been fulfilled, the switch variable changes to state 1 and the power limitation mode is triggered. Otherwise, if  $V < (V_n - \varepsilon)$ , the switching variable passes to State 2, and the TST activates the variable speed operation mode. While the mean value of the tidal speed in an interval of length  $T_{avr}$  remains within the range  $|(V_n - \varepsilon), (V_n + \varepsilon)|$ , the switch controller maintains the previous mode of operation.

The flowchart of the implemented switching control is shown in Figure 15. Indeed, when the tidal speed is in the boundary of the two regions of operation (variable speed and power limitation modes), the system can switch between these modes quite often, leading to extra stress of the pitch actuator. Therefore, the average of the tidal speed is calculated over a time period  $T_{avr}$ , the length of this period being the minimum time where the switch between modes can occur, to set a limit on how frequently the system crosses between operating modes.

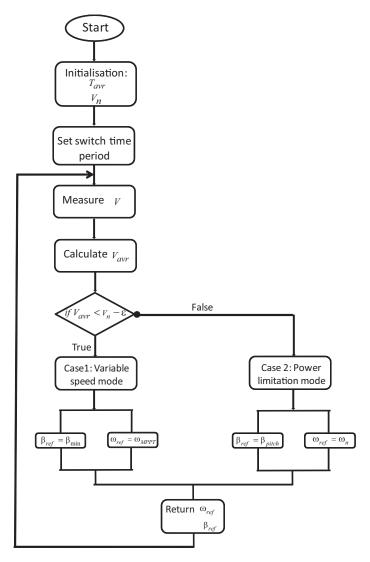


Figure 15. Flowchart of the switching algorithm.

## 5. Results and Discussion

In this section, we will present three case studies to investigate the proposed control strategies. The input considered of the DFIG-based TST system is the tidal current speed, which is chosen in order to study the TST in different modes of operation. Recalling Figure 2 of Section 2, it may be observed that the given stream speed ranges allow testing and comparing the proposed control strategies for the different possible tidal scenarios. We used the system parameters listed in Table 1 in all cases.

**Table 1.** System parameters.

Turbine	Drive-Train	DFIG	Converter
$\rho = 1027 \text{ kg/m}^3$	$H_t = 3 \text{ s}$	$P_n = 1.5 \text{ MW}$	$V_{dc} = 1150 \text{ V}$
R = 8  m	$H_{ m g}=0.5~{ m s}$	$U_{rms} = 690 \text{ V}$	C = 0.01  F
$C_{p  \text{max}} = 0.44$	$K_{sh} = 2 \times 10^6 \text{ Nm/rad}$	$f_{req} = 50 \text{ Hz}$	
$\dot{\lambda}_{opt} = 6.96$	$D_{sh} = 3.5 \times 10^5  \mathrm{Nms/rad}$	$R_s = 2.63 \text{ m}\Omega$	
$V_n = 3.2 \text{ m/s}$		$R_r = 2.63 \text{ m}\Omega$	Choke
		$L_s = 0.168 \text{ mH}$	$R_g = 0.595 \text{ m}\Omega$
		$L_r = 0.133 \text{ mH}$	$L_g^0 = 0.157 \text{ mH}$
		$L_m = 5.474 \text{ mH}$	
		p = 2	

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#### 5.1. Case 1: Low Tidal Speed (Variable Speed Operation)

In this case, variable speed operation is achieved using the rotational speed control strategy. The turbine is operating below rated flow speeds; the rotor speed is being varied to maintain the optimal value of  $C_p$ ; and the pitch angle was set to zero. Figure 16 shows that the turbine is initially operated at a tidal speed of 2 m/s.

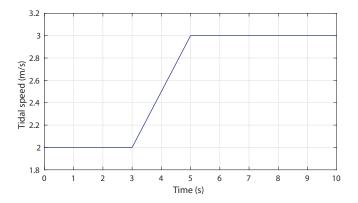
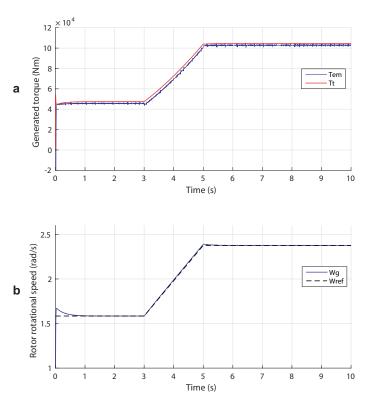


Figure 16. Tidal speed input: Case Study 1.

At the simulation time t=3 s, we mark a step change in the flow speed, which is applied from 2 m/s to 3 m/s. This causes a sudden variation in the hydrodynamic torque, which also has an impact on the generator torque, as shown in Figure 17a. The generated torque achieves a high value of  $1.03 \times 10^5$  Nm for V=3 m/s and a value of  $4.58 \times 10^4$  Nm in the lower case when V=2 m/s. The rotor rotational speed increases according to the flow speed input, as shown in Figure 17b. The control system performs well; thus, the rotational speed is able to track the reference signal generated from the MPPT block.



**Figure 17.** Case Study 1: (a) responses of the hydrodynamic and generator torques to a step change in tidal speed; (b) rotor rotational speed curve and its reference.

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The purpose of this simulation case is to generate the maximum power from the turbine and transfer it to the grid. According to Figure 9, the MPPT control strategy works perfectly. Indeed, as may be seen in Figure 18a, the TST system is capable of maximizing the power output with 350 kW when V = 2 m/s, then increases at t = 3 s to 1.2 MW when V = 3 m/s. Besides, the reactive power, as depicted in Figure 18b, is maintained at zero for unity power factor.

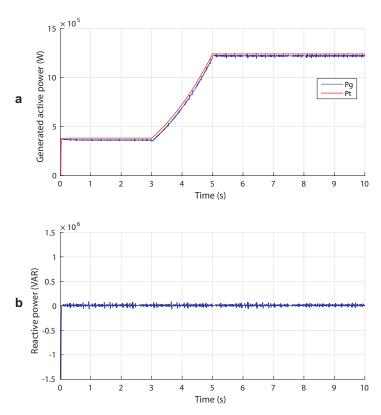


Figure 18. Case Study 1: (a) generated turbine and generator power curves; (b) reactive power curve.

# 5.2. Case 2: High Tidal Speed (Power Limitation Mode)

In this case study, the pitch angle control is tested within the power limitation mode, which imposes an average tidal velocity over  $V_n = 3.2$  m/s. The tidal speed is stepped up from  $V_n$  to V = 3.6 m/s at t = 3 s as shown in Figure 19.

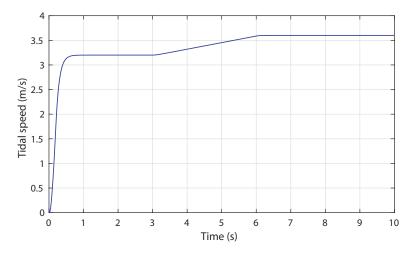


Figure 19. Tidal speed input: Case Study 2.

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The blade pitch angle  $\beta$  should increase to prevent the rotor speed from becoming too high, which would result in alleviation of intense mechanical loads, and consequently, the power coefficient should decrease as shown in Figure 20a,b. The responses of  $C_p$  and  $\beta$  begin with a value of 0.44 and 0 deg, respectively, when  $V < V_n - \varepsilon$ . Then, they achieve a value of 0.31° and 7.79°, respectively, when  $V > V_n + \varepsilon$ . The resulting hydrodynamic and electromagnetic torques are presented in Figure 21a. The value of the generated torque is limited to a value of 6.01 × 10<sup>5</sup> Nm when V exceeds the threshold value. The rotor rotational speed is maintained constant below the limit speed according to the reference signal from the switch controller as depicted in Figure 21b. The figure shows good tracking performances of the rotor speed.

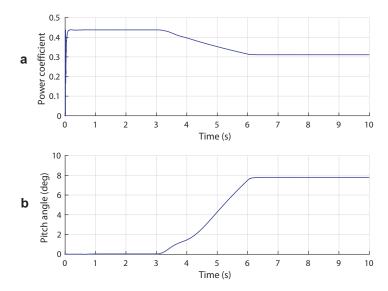
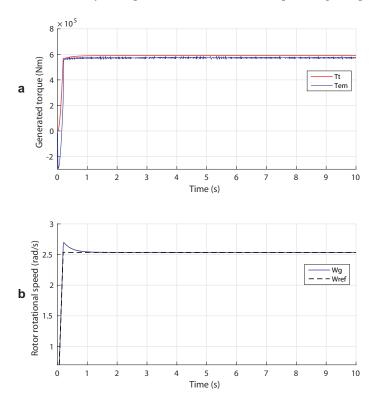


Figure 20. Case Study 2: (a) power coefficient curve; (b) pitch angle response.



**Figure 21.** Case Study 2: (a) responses of the hydrodynamic and generator torques; (b) rotor rotational speed curve and its reference.

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Thus, as Figure 22a shows, the extracted power rises with the tidal step, then it is maintained at the rated power  $P_n$  as the pitch angle  $\beta$  augments. The reactive power is kept oscillating around zero regardless of the change in tidal speed input, as shown in Figure 22b. We note that specifying a rated flow speed, at which the turbine produces its peak power and shedding power at low speeds in excess of the rated value, will increase the capacity factor of the turbine and reduce the cost per kWh of electricity generated.

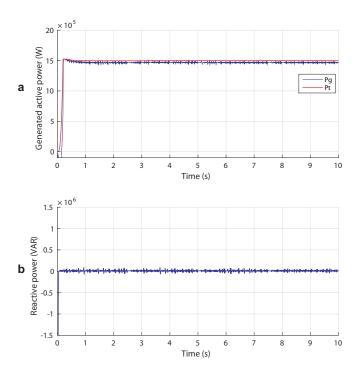


Figure 22. Case Study 2: (a) generated turbine and generator power curves; (b) reactive power curve.

# 5.3. Case 3: Broad Tidal Speed Range (Complementary Control)

This case study examines the novel switching control between the functioning in variable speed and power limitation modes. The shape of the tidal speed input used in this case is depicted in Figure 23. Tidal speed steps from 2.8 m/s up to 3.5 m/s and then down again.

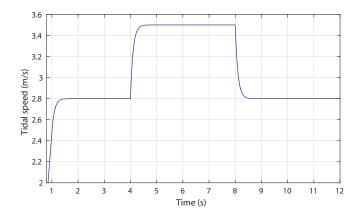


Figure 23. Tidal speed input: Case Study 3.

According to Figure 24, it can be concluded that the switching controller performs well because the power coefficient is maintained at its optimal value ( $C_p = 0.44$ ) and then decreases as shown in Figure 24a. To the contrary, the pitch angle is kept null in the variable speed operation and rises

accordingly to reach a value of the angular position of approximately ( $\beta_{ref} = 5.5^{\circ}$ ) in the case of power limitation mode, as illustrated in Figure 24b.

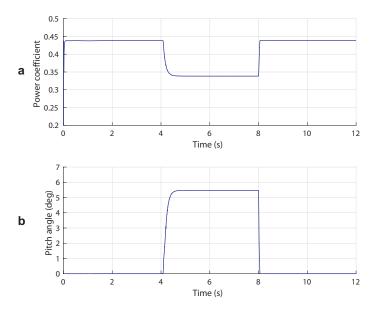
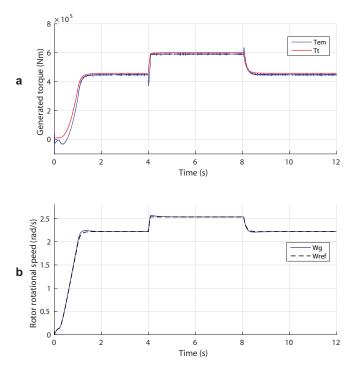


Figure 24. Case Study 3: (a) power coefficient curve; (b) pitch angle response.

The resulting hydrodynamic and generator torques are shown in Figure 25a. The generated torque admits a value of  $4.57 \times 10^5$  Nm in the case of variable speed mode and a value of  $6.01 \times 10^5$  Nm when V exceeds the tolerable value. Figure 25b depicts the rotor rotational speed, which is following the MPPT strategy to track the adequate rotational speed so that the maximum power is obtained. The rotational speed is kept constant to limit the power above nominal tidal speed.



**Figure 25.** Case Study 3: (a) responses of the hydrodynamic and generator torques; (b) rotor rotational speed curve and its reference.

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The expected operating regions are labeled in Figure 26a which illustrates the generated power. In both the first and third regions, where the variable speed mode is used, the power is maximized to 955.1 kW; whereas, in the second region, the variable pitch control is set limiting the maximum power to 1.43 MW. However, the switching within the two modes does not occur immediately due to the time response of the switch controller. We note that the power increases below the rated tidal speed, but it decreases, at high tidal speed, as the rate limiter forces  $\beta$  to increase. The blades take about 0.5 s to reach the desired value. The reactive power transferred to the grid, as shown in Figure 26b, is held around zero.

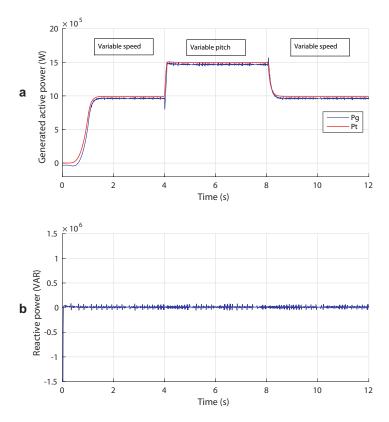


Figure 26. Case Study 3: (a) generated turbine and generator power curves; (b) reactive power curve.

#### 6. Conclusions

This paper dealt with the modeling and control of a DFIG-based TST system connected to the grid through a back-to-back power converter. The common problem with TST is the low power generated at low tidal speed and the excess power at high tidal speed. In this sense, two control strategies were proposed for both cases, and a novel complementary control combining both approaches was implemented.

When the flow speed is below the rated value, a control scheme that maximizes the power output of the TST by allowing the rotor speed to be varied has been developed and evaluated. Stator flux-oriented control was used for the RSC, and variable-speed operation of the rotor is achieved using the generator rotational speed. An MPPT strategy is used to track the optimum rotational speed in order to achieve the maximum power. On the grid-side, the voltage-oriented control scheme was used. The performance of the system was successfully implemented. When tested for a step change in the tidal speed, the RSC adequately varies the rotational speed, improving power extraction, whereas the GSC regulated the flow of active power to the grid, as well as controlling the flow of reactive power. Above the rated tidal current speed, a pitch angle control was investigated to limit the generated power. The switching control was tested among the operations in the variable-speed and power limitation modes. It may be seen that this complementary control presents an improved performance since it prevents frequent transitions when the power generated is at the limits of the two operation modes. Furthermore, the sensitivity of the proposed control strategies was analyzed regarding different tidal

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speed ranges. The controllers, designed to satisfy a predefined set of criteria, provide a satisfactory performance and are able to optimize the power output from the tidal stream generator system.

Three case studies were proposed to test the performance of both control strategies separately, then combined. The obtained results show that the proposed controls provide output power performance improvement in different operating modes.

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